

# Heat Load on a Pillbox Cavity

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## Abstract

Derivation of the power distribution from RF fields on the inner surfaces of a pillbox cavity made of multiple materials.

## Introduction

Consider a cylindrical pillbox cavity with radius  $a$  and length  $L$ . The electromagnetic field amplitudes in SI units are given in cylindrical coordinates  $(r, \phi, z)$  by

$$\begin{aligned}\vec{E}(r) &= E_0 J_0(j_{01} \frac{r}{a}) \hat{z} \\ \vec{B}(r) &= \frac{E_0}{c} J_1(j_{01} \frac{r}{a}) \hat{\phi}\end{aligned}$$

in terms of Bessel functions of the first kind  $J_0, J_1$  with

$$\begin{aligned}\omega &= c \frac{j_{01}}{a} = 2\pi f \\ J_0(j_{01}) &= 0\end{aligned}$$

where

- $E_0$  is the peak on-axis gradient
- $c \simeq 2.9979 \times 10^8$  m/s is the speed of light,
- $f$  is the resonant frequency of the cavity and
- $j_{01} \simeq 2.4048$  is the smallest root of  $J_0$

## Stored energy

The electromagnetic energy density in the cavity volume is

$$w = \frac{1}{2} \left[ \epsilon_0 E(r,t)^2 + \frac{1}{\mu_0} B(r,t)^2 \right]$$

and with the full time-dependent fields expressed as

$$\begin{aligned} \vec{E}(r,t) &= \vec{E}(r) \cos \omega t \\ \vec{B}(r,t) &= \vec{B}(r) \sin \omega t \end{aligned}$$

the total stored energy can be found when  $\omega t = 0$  or  $\pi$  through

$$U = \epsilon_0 L \int_0^a dr 2\pi r E^2(r) = \frac{L}{\mu_0} \int_0^a dr 2\pi r B^2(r)$$

using

$$\int_0^1 dx x J_0^2(j_{01} x) = \int_0^1 dx x J_1^2(j_{01} x) = \frac{1}{2} J_1^2(j_{01})$$

as

$$U = J_1^2(j_{01}) (\pi a^2 L) \left[ \frac{1}{2\mu_0} \left( \frac{E_0}{c} \right)^2 \right]$$

where  $J_1^2(j_{01}) \simeq 0.26951$  showing that the (spatial) average energy density is about 27% of the peak in this simple structure and does not depend on the aspect ratio.

## Surface currents and power dissipation

In a cavity made of material with finite conductivity, power dissipation is due to a thin layer of current under the surface confined to the skin depth  $\delta$ . The current density  $\vec{J}$  is given by

$$\begin{aligned} \vec{J}_{cr} &= \frac{B_\phi(a)}{\mu_0 \delta_{cr}} \hat{\mathbf{z}}, \quad -\frac{L}{2} < z < \frac{L}{2} \\ \vec{J}_{e1} &= \frac{B_\phi(r)}{\mu_0 \delta_{e1}} \hat{\mathbf{r}}, \quad z = -\frac{L}{2} \\ \vec{J}_{e2} &= \frac{B_\phi(r)}{\mu_0 \delta_{e2}} \hat{\mathbf{r}}, \quad z = \frac{L}{2} \end{aligned}$$

where

- $\mu_0 = 4\pi \times 10^{-7}$  H/m is the permeability of vacuum and
- the subscripts cr, e1 and e2 refer to the center ring and the two endplates respectively.

The corresponding (rms) volume and surface power density can be written as

$$\begin{aligned}\frac{dP}{dV} &= \frac{1}{2} \frac{1}{\sigma} J^2 \\ \frac{dP}{dA} &= \delta \frac{dP}{dV} = \frac{1}{2} \frac{\delta}{\sigma} J^2\end{aligned}$$

where  $\sigma$  is the conductivity. This yields

$$\begin{aligned}\frac{dP_{cr}}{dA} &= \frac{1}{2} \frac{B_\phi^2(a)}{\mu_0^2 \delta_{cr} \sigma_{cr}} = \frac{1}{2} R_{cr} \left[ \frac{E_0}{\mu_0 c} J_1(j_{01}) \right]^2 \\ \frac{dP_{e1}}{dA} &= \frac{1}{2} \frac{B_\phi^2(r)}{\mu_0^2 \delta_{e1} \sigma_{e1}} = \frac{1}{2} R_{e1} \left[ \frac{E_0}{\mu_0 c} J_1(j_{01} \frac{r}{a}) \right]^2 = \frac{R_{e1}}{R_{cr}} \left[ \frac{J_1(j_{01} \frac{r}{a})}{J_1(j_{01})} \right]^2 \frac{dP_{cr}}{dA} \\ \frac{dP_{e2}}{dA} &= \frac{1}{2} \frac{B_\phi^2(r)}{\mu_0^2 \delta_{e2} \sigma_{e2}} = \frac{1}{2} R_{e2} \left[ \frac{E_0}{\mu_0 c} J_1(j_{01} \frac{r}{a}) \right]^2 = \frac{R_{e2}}{R_{e1}} \frac{dP_{e1}}{dA}\end{aligned}$$

with the surface (sheet) resistance  $R$ , skin depth and conductivity related through

$$\delta \simeq \sqrt{\frac{2}{\mu_0 \omega \sigma}}, \quad R = \frac{1}{\delta \sigma} \simeq \sqrt{\frac{\mu_0 \omega}{2 \sigma}}$$

The total heat deposited on the surface follows from integration of these as

$$\begin{aligned}P_{cr} &= (2\pi a L) \frac{dP_{cr}}{dA} = \pi a L R_{cr} \left[ \frac{E_0}{\mu_0 c} J_1(j_{01}) \right]^2 \\ P_{e1,2} &= \frac{R_{e1,2}}{R_{cr}} \frac{a}{2L} P_{cr} \\ P &= P_{cr} + P_{e1} + P_{e2}\end{aligned}$$

## Quality factor

Using the expressions for  $U$  and  $P$  above, the quality factor of the cavity resonator is found as

$$Q = \omega \frac{U}{P} = \frac{\sigma_{cr} R_{cr} a}{1 + \frac{R_{e1} + R_{e2}}{2 R_{cr}} \frac{a}{L}} = \frac{a}{\delta_{cr}} \frac{1}{1 + \frac{\delta_{e1} + \delta_{e2}}{2 \delta_{cr}} \frac{a}{L}}$$

We can express this in terms of the individual components as

$$\frac{1}{Q} = \frac{1}{Q_{cr}} + \frac{1}{Q_{e1}} + \frac{1}{Q_{e2}}$$

where

$$Q_{cr} = \frac{\omega U}{P_{cr}} = \frac{a}{\delta_{cr}}, \quad Q_{e1,2} = \frac{\omega U}{P_{e1,2}} = \frac{2L}{\delta_{e1,2}}$$

## Effect of the interface

Consider additional power dissipation  $P_g$  and corresponding  $Q_g$  at the interfaces (eg. gaskets) between the endplates and the center ring. The resulting  $Q$  is

$$\frac{1}{Q} = \frac{1}{Q_{cr}} + \frac{1}{Q_{g1}} + \frac{1}{Q_{e1}} + \frac{1}{Q_{g2}} + \frac{1}{Q_{e2}}$$

Dissipation at the interface is not easy to model. If the endplates and interfaces are the same on both sides, the unknown  $Q_g$  can be calculated using the measured cavity  $Q$  and the known surface properties.

## Effect of coating

A thin layer of TiN is often used as a multipacting barrier. This leads to additional power dissipation on the surface. For a layer of thickness  $t$  much less than the skin depth  $\delta_t$ , the equivalent surface resistance would be

$$R_t = \frac{1}{\sigma_t} \frac{t}{\delta_t}$$

If placed over a substrate with power dissipation  $P_s$ , the additional power consumed by the TiN layer can be expressed as

$$\frac{dP_t}{dA} = \frac{R_t}{R_s} \frac{dP_s}{dA}$$

and the effective  $Q$  of the surface changes accordingly

$$Q_s \rightarrow \left(1 + \frac{R_t}{R_s}\right)^{-1} Q_s$$

## Numerical example

Consider the MTA Modular Cavity with

$$\begin{aligned}a &= 14.1 \text{ cm} \\L &= 10.3 \text{ cm} \\f &= 805 \text{ MHz}\end{aligned}$$

The center ring is made of copper and endplates made of both copper and beryllium are available.

$$\begin{aligned}\sigma_{Cu} &= 6.0 \times 10^7 (\Omega\text{m})^{-1} \\ \frac{\sigma_{Be}}{\sigma_{Cu}} &= 0.43 \\ \frac{\sigma_{TiN}}{\sigma_{Be}} &= 0.10 \\ \frac{t}{\delta_t} &= 0.01\end{aligned}$$

The table below shows various quantities to aid in exploring different endplate configurations. Cavity Q refers to the unloaded quality factor. No correction for the TiN coating is included.

Theory		
Peak energy density		$[4.42 \text{ (J/m}^3\text{)/(MV/m)}^2] E_0^2$
Average energy density		$[1.19 \text{ (J/m}^3\text{)/(MV/m)}^2] E_0^2$
Stored energy	$U$	$[7.65 \text{ mJ/(MV/m)}^2] E_0^2$
Assumed		
Skin depth (Cu)	$\delta_{Cu}$	2.29 $\mu\text{m}$
Skin depth (Be)	$\delta_{Be}$	3.49 $\mu\text{m}$
Surface resistance (Cu)	$R_{Cu}$	7.28 $\text{m}\Omega$
Surface resistance (Be)	$R_{Be}$	11.1 $\text{m}\Omega$
Surface resistance (TiN)	$R_{TiN}$	0.35 $\text{m}\Omega$
Calculated		
Center ring power	$P_{cr}$	$[630 \text{ W/(MV/m)}^2] E_0^2$
Endplate power (Cu)	$P_e$	$[431 \text{ W/(MV/m)}^2] E_0^2$
Endplate power (Be)	$P_e$	$[658 \text{ W/(MV/m)}^2] E_0^2$
Cavity power (2 Cu ep, no i/f)	$P$	$[1.49 \text{ kW/(MV/m)}^2] E_0^2$
Cavity power (1 Cu + 1 Be ep)	$P$	$[1.72 \text{ kW/(MV/m)}^2] E_0^2$
Cavity power (2 Be ep)	$P$	$[1.95 \text{ kW/(MV/m)}^2] E_0^2$
Component Q (center ring)	$Q_{cr}$	61.6k
Component Q (Cu endplate)	$Q_e$	89.8k
Component Q (Be endplate)	$Q_e$	58.9k
Cavity Q (all Cu, no interface)		26.0k
Cavity Q (1 Cu & 1 Be endplate)		22.5k
Cavity Q (2 Be endplates)		19.9k
Measured		
Measured cavity Q (2 Cu endplates)		21.2k
Measured cavity Q (2 Be endplates)		13.5k
Inferred		
Interface Q (Cu-ep, calculated)	$Q_g$	231k
Interface Q (Be-ep, calculated)	$Q_g$	83.8k
Interface power (Cu-ep)	$P_g$	$[168 \text{ W/(MV/m)}^2] E_0^2$
Interface power (Be-ep)	$P_g$	$[463 \text{ W/(MV/m)}^2] E_0^2$
Predicted		
Expected cavity Q (1 Cu & 1 Be ep)		16.5k
Cavity power (Cu+Cu ep)	$P$	$[1.83 \text{ kW/(MV/m)}^2] E_0^2$
Cavity power (Be+Cu ep)	$P$	$[2.35 \text{ kW/(MV/m)}^2] E_0^2$
Cavity power (Be+Be ep)	$P$	$[2.87 \text{ kW/(MV/m)}^2] E_0^2$