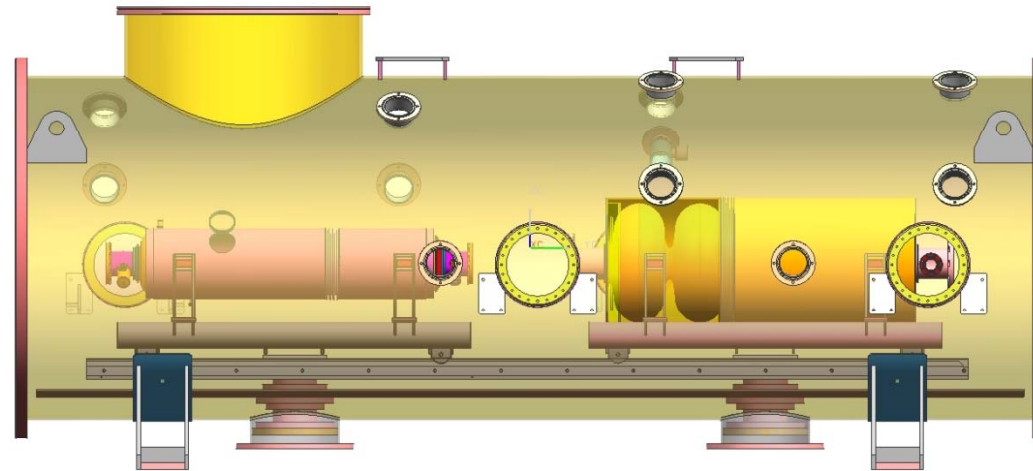


Horizontal Test Stand -2 Development Effort at RRCAT

A 3-D Model as
developed by
RRCAT



**P K Kush,
P Khare, S Gilankar, R Ghosh, A Jain**

- **Introduction**
- **Status of the design effort**
- **Path to be traversed**
- **Details of the work carried out so far in**
 - A. Vacuum Vessel**
 - B. Cavity Support system**
 - C. Thermal Shields**
 - D. Cryogenic Circuit**
 - E. Glimpses of the 3-D model developed in UGNX-4**
- **Queries related with this system so far.**
- **Conclusion**

- HTS-2 has to be designed and developed at RRCAT for testing of fully dressed SCRF cavities: 1.3GHz and 650MHz in both CW and pulsed mode.
- **Fermilab has expressed that**
“With Fermi Lab’s help, RRCAT will fabricate a minimum of two cryostats, one of which will be delivered to Fermilab (Q2 of 2012 preferably or end of Q4-2012)”.
- **The work has been initiated to design a cryostat of HTS-2. Some glimpses are given in next few slides.**
- There are certain dark areas and also there are certain queries in the areas that we are working, which are also presented.

- Design efforts were initiated in 2009 to design and develop HTS-2.
 - A functional requirement specification (FRS) draft document was prepared and sent to FNAL in Jan/Feb 2010.
 - After a pause of about 6 months, FRS document was received by us in August 2010.
 - Without waiting for other documents like Technical Requirement Specification and Technical Design Report we have initiated design of following subsystems
 - A. Vacuum Vessel
 - B. Cavity Support system
 - C. Thermal Shields
- } All are Related to cryostat to be supplied
- The idea is to cover some ground till we receive detailed reports and then modify the drawings/model accordingly. Thus we intend to save some time.
 - With the same thought a complete 3-D model has been completed for HTS-2 which gives us an insight into the internal configuration and sizing of components.

Path to be traversed

Instead of waiting for the initiation of Technical requirement Specification.

1. We performed literature survey of similar facilities
2. Studied the drawings of HTS-1
3. Prepared a model to visualize the mechanical arrangement
4. Started working on designing the subsystems. The idea is to prepare drawings/models/calculations & if technical design report indicates changes then we will incorporate those changes.

Objective was to minimize the time required

- Like to receive answers to queries, enlisted in further slides.
- The effort has now to begin on
 - A. RF powering
 - B. Instrumentation and Controls



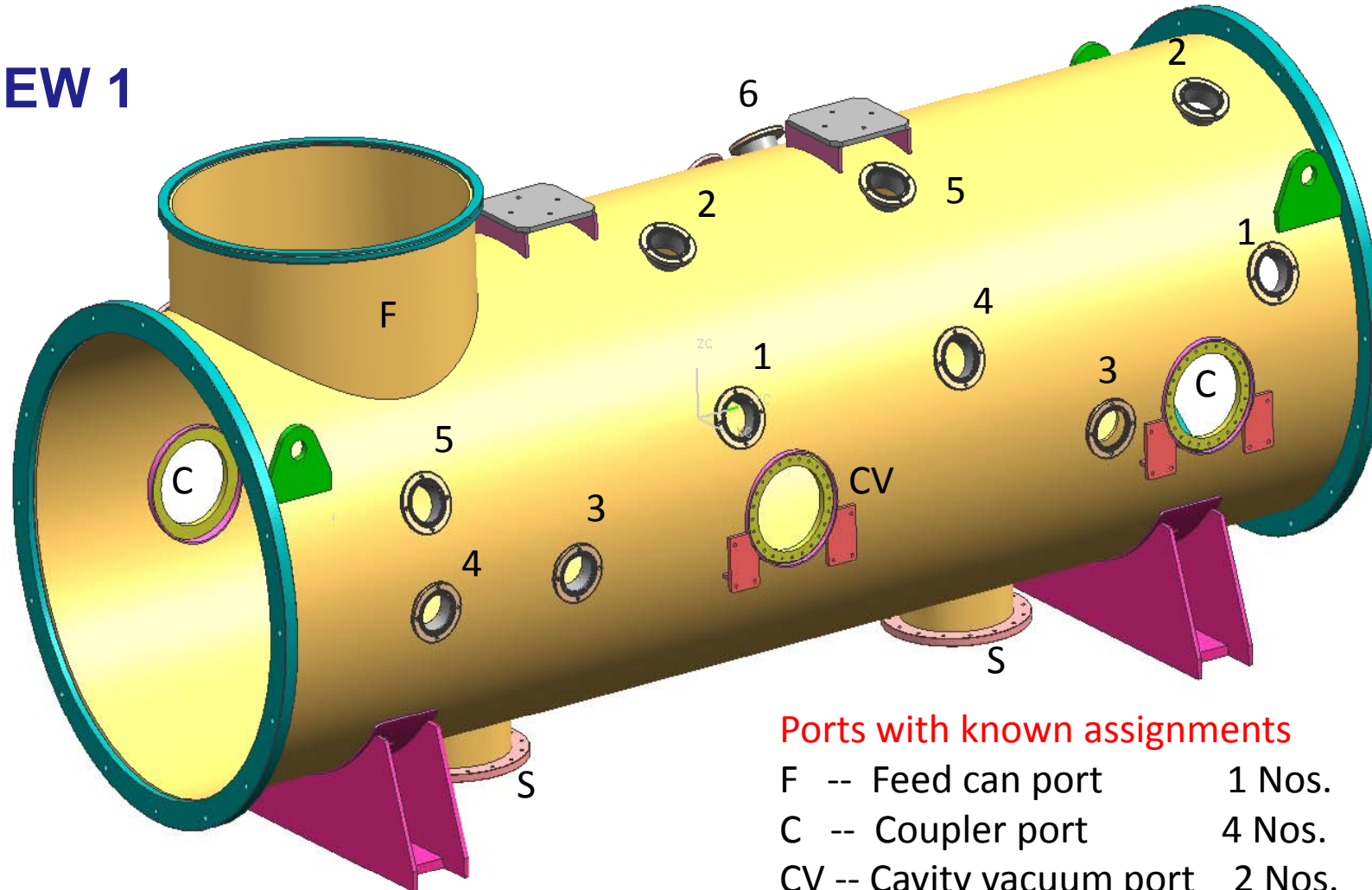
Details of Works Carried out



Specifications assumed for the Initiating the Task

Parameter	Value
Mechanical	
Interior usable length	3.5 m
Interior diameter	1.2 m
Option for testing number of cavities	Two 9 cell cavities or operate as a cryomodule
Main Cryogenics	
Temperature	1.8 – 2.2 K, 4.2 K
Capacity	90 W/ 170 W (1.8 K/4.2 K)
Pressure Stability	± 0.1 mbar/1 mbar (1.8 K/4.2K)
Static losses	12 mW at 1.8 K (with 4 K shield) (without load of coupler and instrumentation leads)
Secondary Loop	
Table cooling loop	4.5 K
Cavity fill	4.5 K
Coupler cooling	4.5 - 20 K
Radiation shield	77 K
Vacuum Pumps	
Suction pressure	14 mbar
Speed	9000 m ³ /hr
RF Power	
Frequency	1.3 GHz
Power	10 kW

VIEW 1



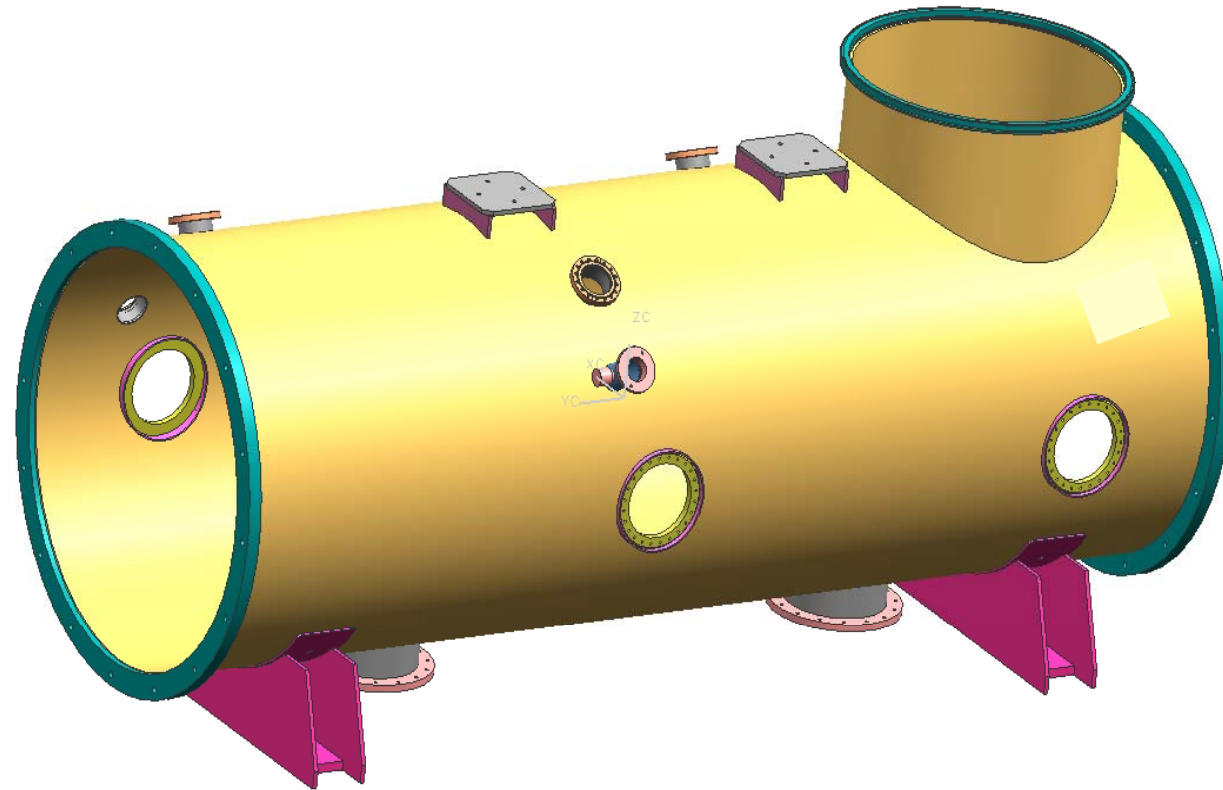
Ports with known assignments

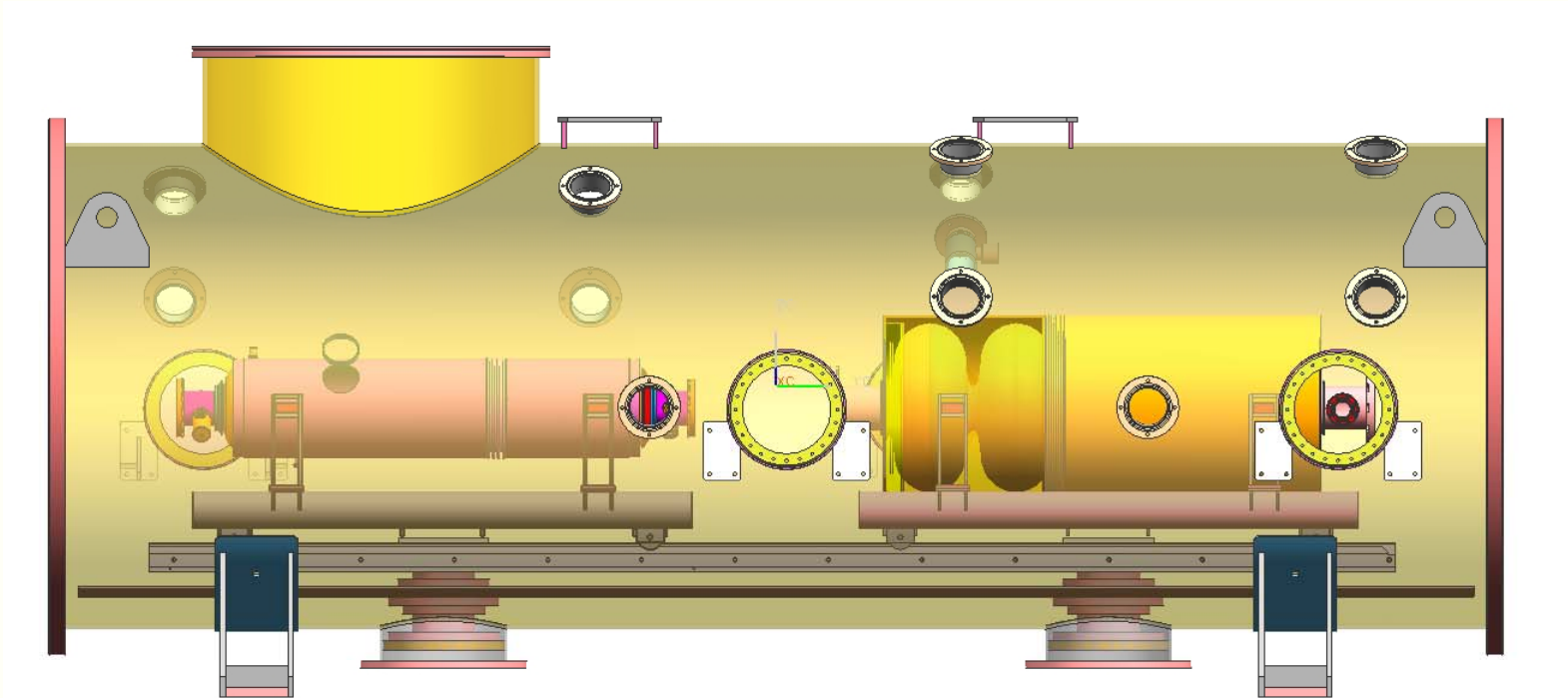
F -- Feed can port	1 Nos.
C -- Coupler port	4 Nos.
CV -- Cavity vacuum port	2 Nos.
S -- Support post port	2 Nos.

HTS-2 Vacuum Vessel (3-D Model in UGNX-4)

- Developed at RRCAT

VIEW 2





3-D Model Completed in UGNX-4 at RRCAT

Status

- A. Internal configuration finalized.
- C. Cavity support System Prototyping in Progress.
- D. Initiated thermal shield design.
- E. Vacuum vessel design note -1st draft ready.



Preliminary Vacuum Vessel Parameters



Preliminary vacuum vessel of HTS-2	Designed value	Remark
Diameter of SS 304 Vessel*	46 inch	Considering testing of 650 MHz cavity also
Overall Length of Vessel	137.7 inch	Considering testing of two SCRF cavities
Nominal Thickness of Vessel	0.375 inch	

Verified with ASME Boiler and Pressure Vessel Code Sec.VIII Div 1,(2004)

Description of openings in vessel	Size	Numbers	Remark
Feedcan port	32 inch	1	Size dependant upon final cryogenic feedbox design (port wall thickness 0.312”)
Main RF coupler Port	11.32 inch	4	Number of ports according to way the numbers of cavities tested (port wall thickness 0.187 “)
Support Post Port (bottom)	12 inch	2	(Wall thickness 0.25”)
Other Instrumentation Ports	4 inch	12	Number of ports considering two cavities (Wall thickness 0.12”)
Ports for Vacuum Pumping (Same is used for parallel plate type relief valve ?)	4 inch	1	(Wall thickness 0.12”)
Port for Parallel Plate Relief Valve (with seal off valve ?)	2.88 inch	1	(wall thickness 0.12”)

Note: No external reinforcement is required for any of the above openings



Tentative Assignment of Instrumentation Ports

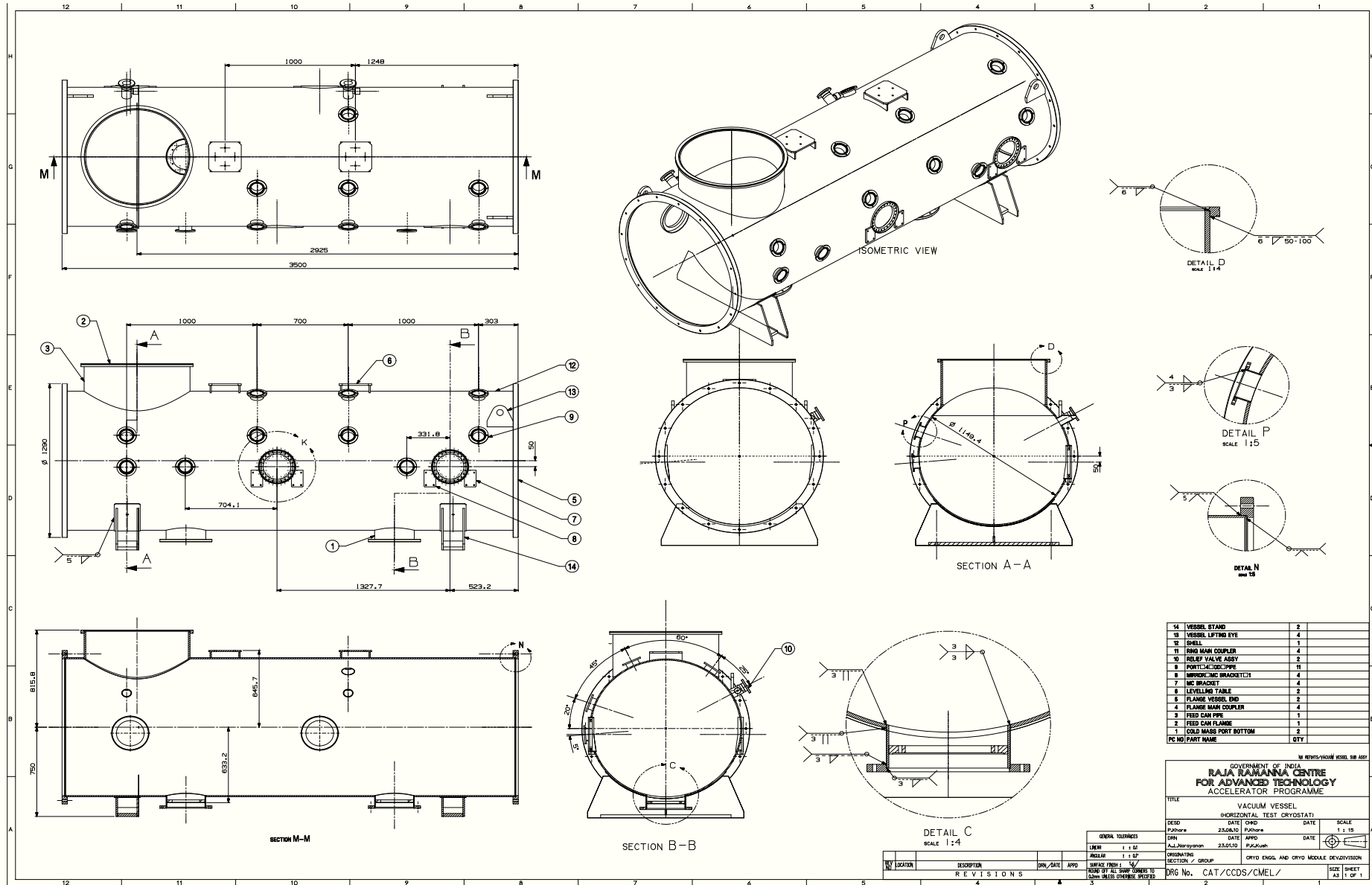
(Referred from documents on internal instrumentation for HTS-1)



Port No.	Size of ports (in)	Identification as	No. of ports	Remarks
1	Ø 4	Coupler and RF	Two	Location ?
2	Ø 4	Farady cup and miscellaneous	Two	Location ?
3	Ø 4	Cryostat Instrumentation	Two	Location ?
4	Ø 4	Fast Tuner	Two	Location of second port due to increased Feedcan port ?
5	Ø 4	Miscellaneous	One	Location of second port due to increased Feedcan port ?
6	Ø 4	Slow Tuner and miscellaneous	One or Two ?	Number of ports and location?
7	Ø 2.88 ?	Parallel plate relief port and vessel pumping port	One or Two ?	

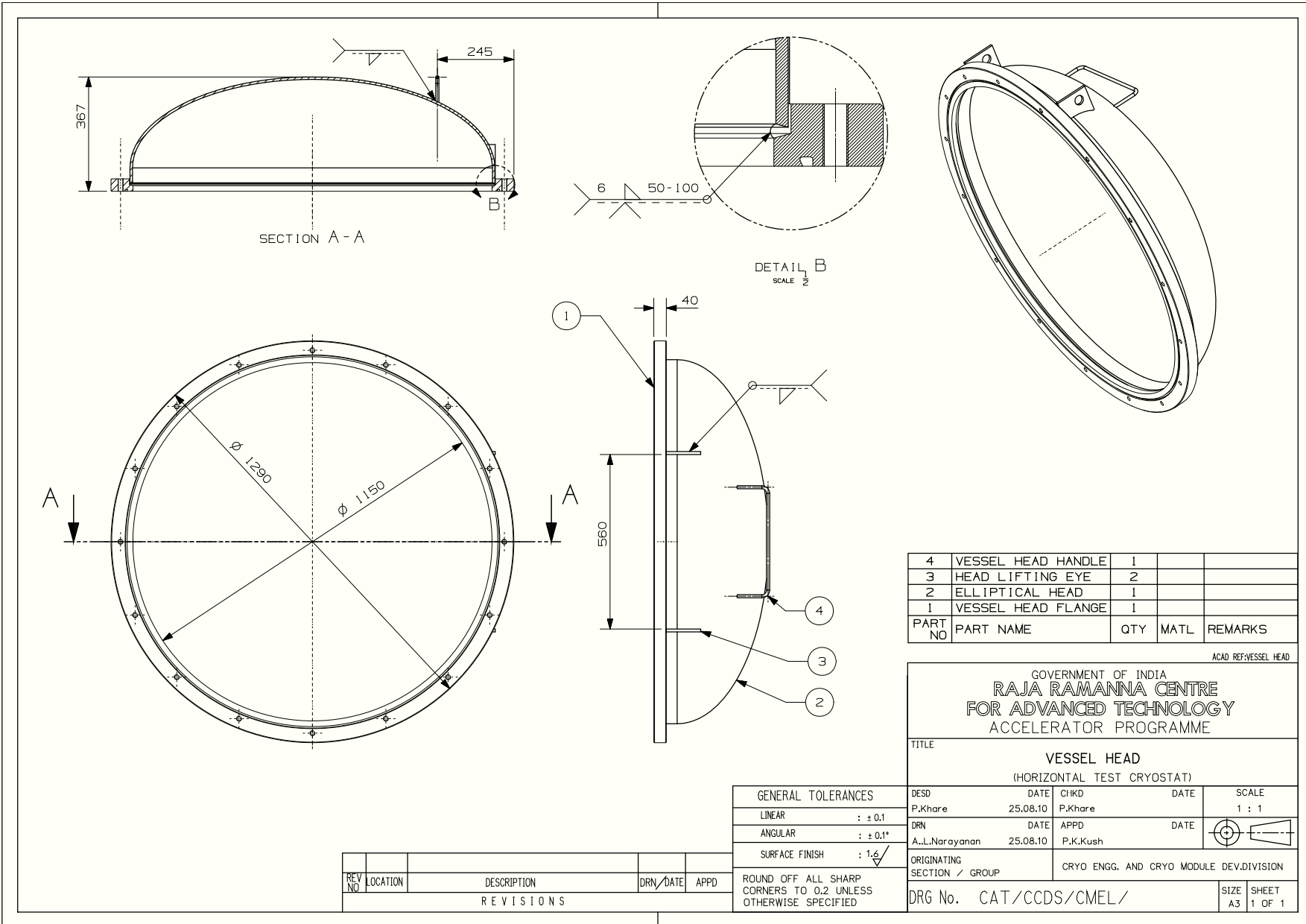
HTS-2 Vacuum Vessel (2-D Drawings)

Being Prepared at RRCAT



HTS-2 Vacuum Vessel (2-D Drawings)

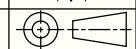
Being Prepared at RRCAT



4	VESSEL HEAD HANDLE	1		
3	HEAD LIFTING EYE	2		
2	ELLIPTICAL HEAD	1		
1	VESSEL HEAD FLANGE	1		
PART NO	PART NAME	QTY	MATL	REMARKS

GOVERNMENT OF INDIA
RAJA RAMANNA CENTRE
 FOR ADVANCED TECHNOLOGY
 ACCELERATOR PROGRAMME

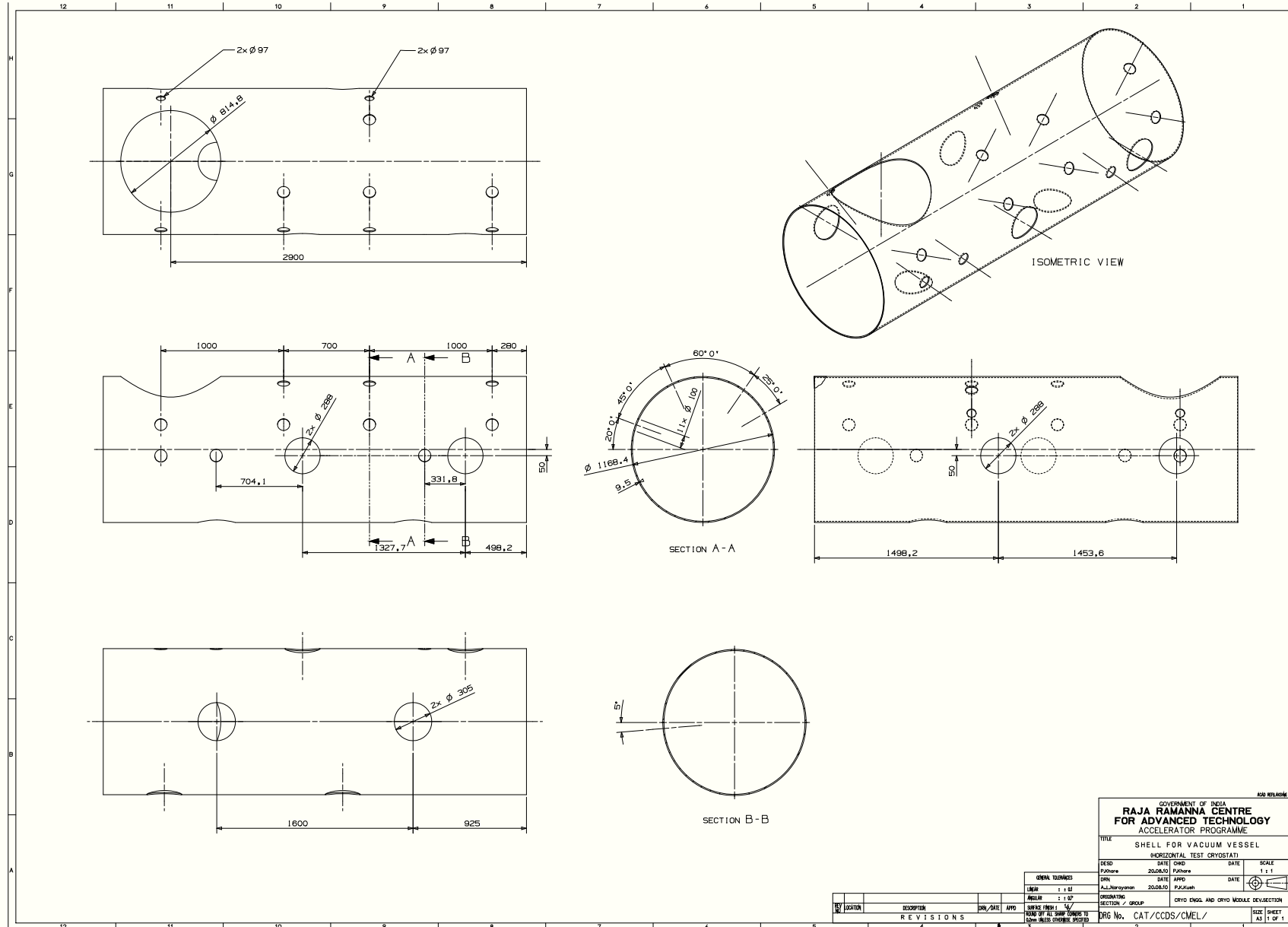
TITLE
VESSEL HEAD
 (HORIZONTAL TEST CRYOSTAT)

GENERAL TOLERANCES		DESIGNER	DATE	CHKD	DATE	SCALE		
LINEAR	: ± 0.1	P.Khare	25.08.10	P.Khare		1 : 1		
ANGULAR	: ± 0.1°	DRN	DATE	APPD	DATE			
SURFACE FINISH	: 1.6/√	A..L.Narayanan	25.08.10	P..K.Kush				
ORIGINATING SECTION / GROUP		CRYO ENGG. AND CRYO MODULE DEV.DIVISION						
REV NO		LOCATION		DESCRIPTION		DRN/DATE	APPD	
REVISIONS								
ROUND OFF ALL SHARP CORNERS TO 0.2 UNLESS OTHERWISE SPECIFIED						DRG No. CAT/CCDS/CMEL/	SIZE A3	SHEET 1 OF 1



HTS-2 Vacuum Vessel (2-D Drawings)

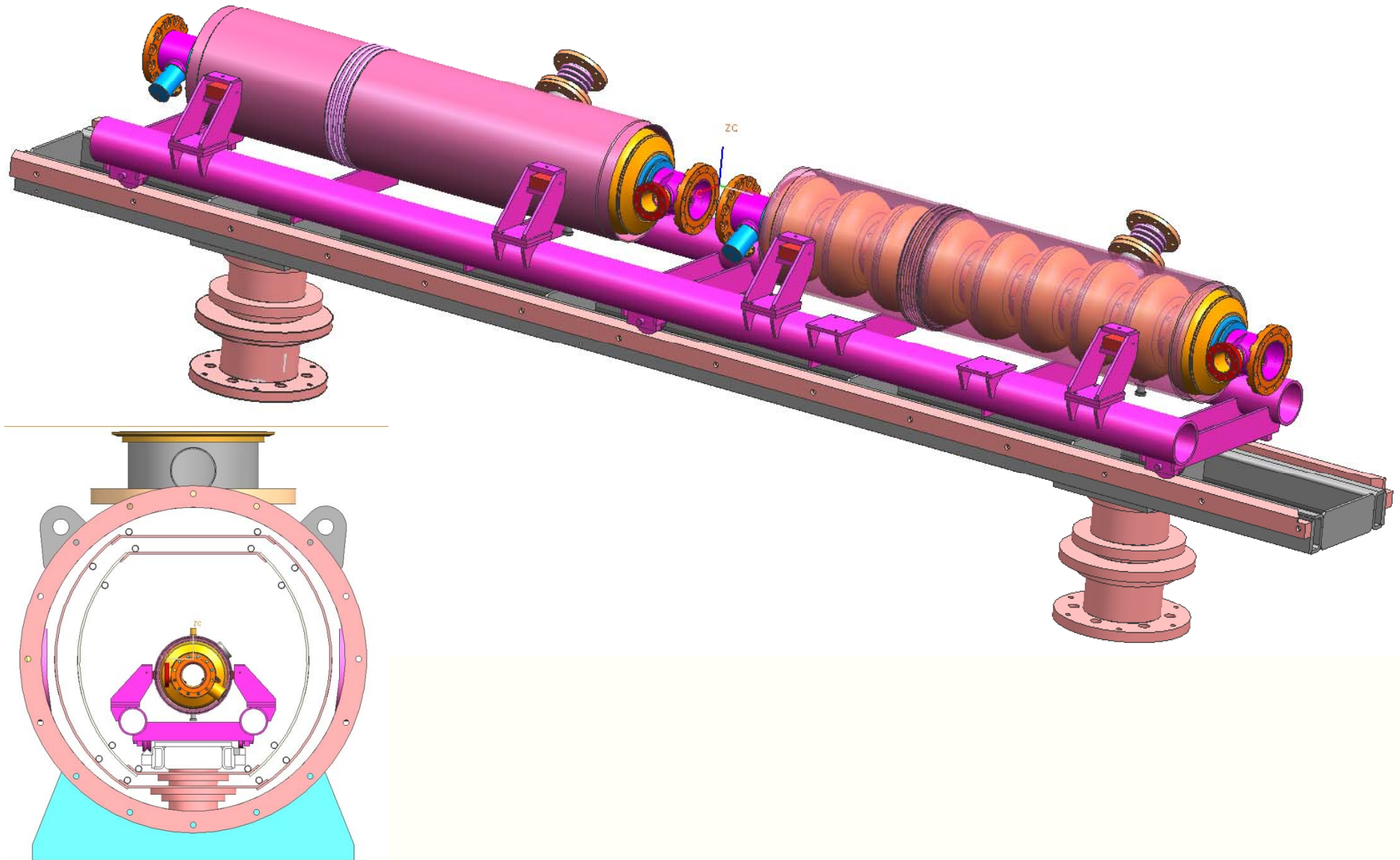
Being Prepared at RRCAT



GOVERNMENT OF INDIA		RAJA RAMANNA CENTRE FOR ADVANCED TECHNOLOGY	
ACCELERATOR PROGRAMME		TITLE	
HORIZONTAL TEST CRYOSTAT		SHELL FOR VACUUM VESSEL	
DESIGN	DATE	SCALE	1 : 1
DESIGNED BY	20.08.20	DESIGNED BY	20.08.20
CHECKED BY	20.08.20	CHECKED BY	20.08.20
APPROVED BY	20.08.20	APPROVED BY	20.08.20
REGISTRATION / GROUP		CRYO ENGL. AND CRYO MODULE DEVELOPMENT	
SECTION / GROUP		CRYO ENGL. AND CRYO MODULE DEVELOPMENT	
PROJECT NO.		CAT/CCDS/CMEL/	
SHEET NO.		1 OF 1	

NO.	DESCRIPTION	DATE	BY
1	ISSUED FOR PREPARE		
2	REVISIONS		

HTS-2 Cavity Support System- 3d model Being Prepared at RRCAT





Cold Support Post

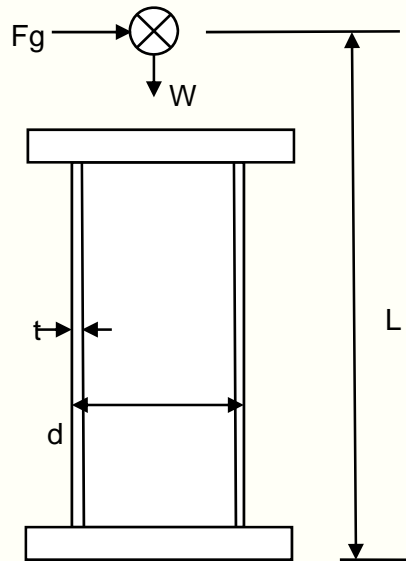


The cold support post of Tesla type IV cryomodules is identical to cryogenic support post design of LHC support post.

We are making an effort to design the cold support post from the first principle. The reason for which is two fold

- a. There is no design available, what is available is just the partial drawings of cryogenic support post. Until and unless we understand how it was designed we will not be able to check cryogenic support post for new load conditions.
- b. We also want to explore whether there is a possibility to reduce the overall length of support post in order to create space for 4K shield in same diameter of vacuum vessel.

Calculate Thickness required for G11 tube



Considering Tensile Stress due to Bending and Direct Compression at the Base

$$\sigma = \frac{F_g L d}{2I} - \frac{W}{A} \quad \Rightarrow \quad t = \frac{1}{\sigma} \left[\frac{4F_g L}{\pi d^2} - \frac{W}{\pi d} \right]$$

Next Check for Elastic Instability

$$\sigma_e = \frac{2Et}{1.5\sqrt{3}\sqrt{1-\mu^2} d}$$

$$t^2 = \frac{1.5\sqrt{3}\sqrt{1-\mu^2}}{2E} \left[\frac{4F_g L}{\pi d} - \frac{W}{\pi} \right]$$

F_g is lateral component of shipping, handling and seismic load

'W' is load per support post

'L' is the distance of C.G. of cold mass from base of support post

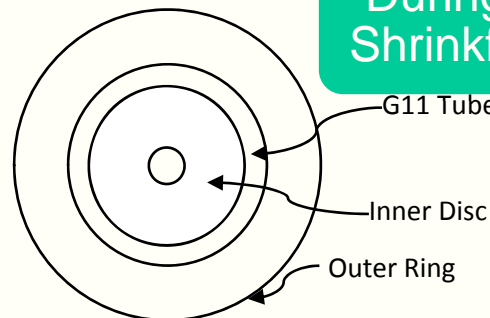
Basic Model for Analyzing Shrink fit Joint of Support Post

Before Assembly

- Interference δ_i between Inner Disc and Tube
- Clearance δ_o between Tube and Outer Ring

During Shrinkfit

- ID of G11 Tube Increases while OD of Disc Decreases to a Common Inner Interface Diameter
- OD of G11 Tube Increases along with Outer Ring to a Common Outer Interface Diameter

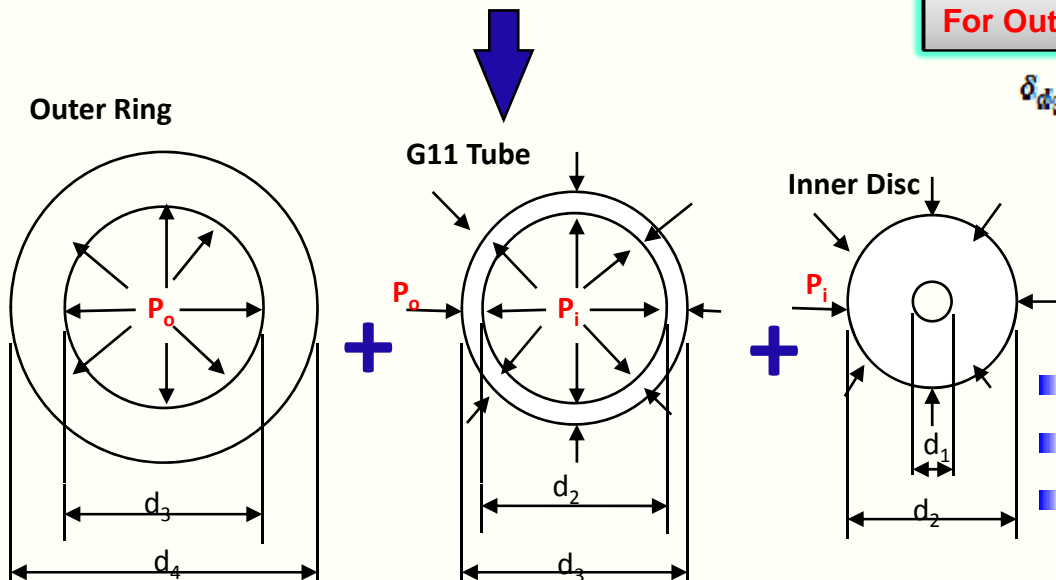


$$ID_{G11} + \delta_{d_2, G11} = OD_{disc} - \delta_{d_2, Disc} \quad \text{For Inner Joint}$$

$$\delta_{d_2, Disc} + \delta_{d_2, G11} = OD_{disc} - ID_{G11} = \delta_i$$

$$\text{For Outer Joint} \quad OD_{G11} + \delta_{d_3, G11} = ID_{Ring} + \delta_{d_3, Ring}$$

$$\delta_{d_3, G11} = ID_{Ring} - OD_{G11} + \delta_{d_3, Ring} = \delta_o + \delta_{d_3, Ring}$$



$\delta_{d_2, Disc}, \delta_{d_2, G11}, \delta_{d_3, G11}, \delta_{d_3, Ring} \Rightarrow$ Change in ID/OD of Disc, Tube and Ring

- Compound Cylinder with Three Components
- Stress Formulae for Thick Cylinders Used to Find δ
- Solve for P_i and P_o from Two above Coupled Equations

Load bearing Capacity for HTS

Result for HTS

P_o =	20.44	N/mm ²
P_i =	21.35	N/mm ²
F_o =	4494	Kg
F_i =	4617	Kg

P_i is Inner Contact Pressure between Disc and Tube
P_o is Outer Contact Pressure between Tube and Ring

F_i is Max Axial Load Carried by Inner Disc
F_o is Max Axial Load Carried by Outer Ring

ID of SS Disc, d1	19mm
ID of G11Tube, d2	149.9mm
OD of G11Tube, d3	152.4mm
OD of SS Ring, d4	178mm
Diametral Interference Disc and Tube, δ_i	0.22mm
Diametral Clearance Ring and Tube, δ_o	0.1mm

Poisson's Ratio SS304 Disc μ₁	0.3
Poisson's Ratio G11 Tube μ₂	0.2
Poisson's Ratio SS304 Ring μ₃	0.3
Young's Modulus SS304 Disc, E₁	200Gpa
Young's Modulus G11 Tube, E₂	28Gpa
Young's Modulus SS304 Ring, E₃	200Gpa

Work to be Performed

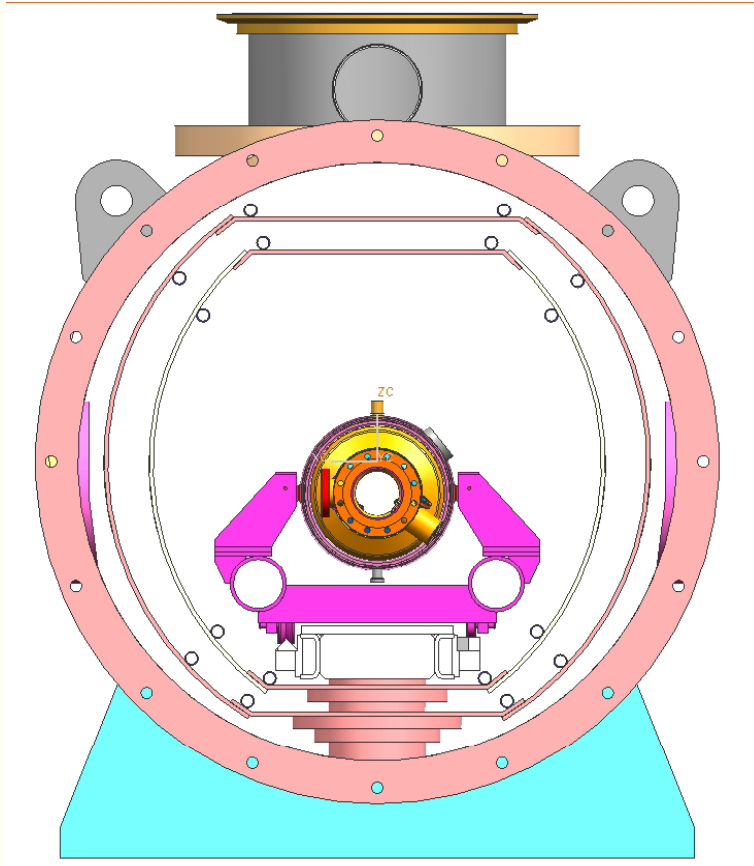
- ❖ Find Optimum Position of 80K and 4K Intercept by Minimizing Total Room Temperature Refrigeration Load
- ❖ Calculate Heat Loads at 80K, 4K and 2K
- ❖ Finite Element Analysis of Support Post Structure

Need Inputs for Completing the analysis

- ❖ Details of Cold Mass Weight and Distribution
- ❖ Details of Shock Loads
- ❖ Estimate Lateral resonant Frequency of Support Post

HTS-2 Thermal Shields - 3d model

Being Prepared at RRCAT

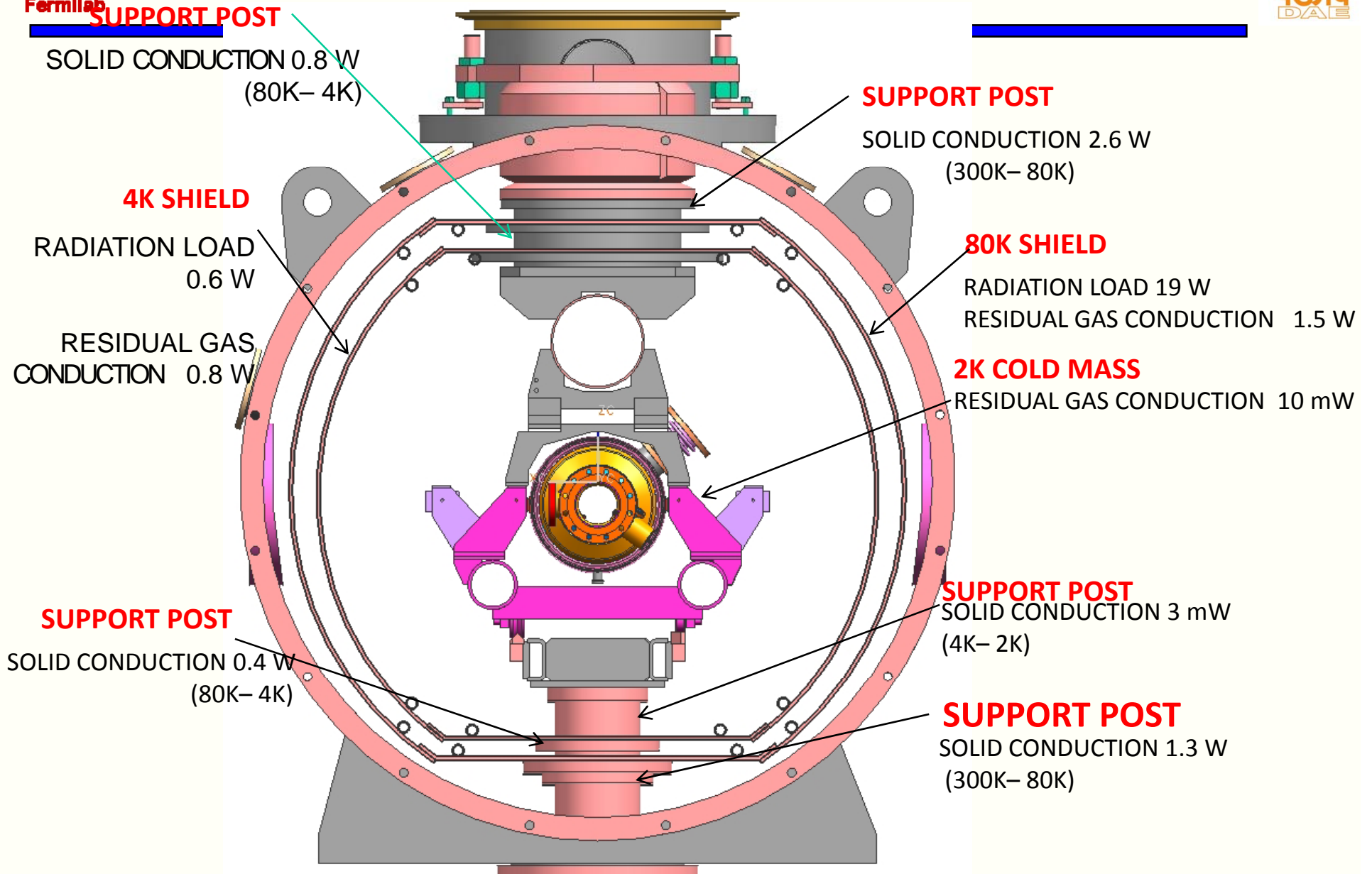


1. Assumed Vessel Dia to be 46 inch
2. Thermal shields 80K and for 5-8K
3. Two three options for layout of cool down pipes being contemplated.
4. We are targeting a figure of 24 hrs for the cool down.



Fermilab

Crosssection & Heat Leak Initial Estimates (CCDS, RRCAT)



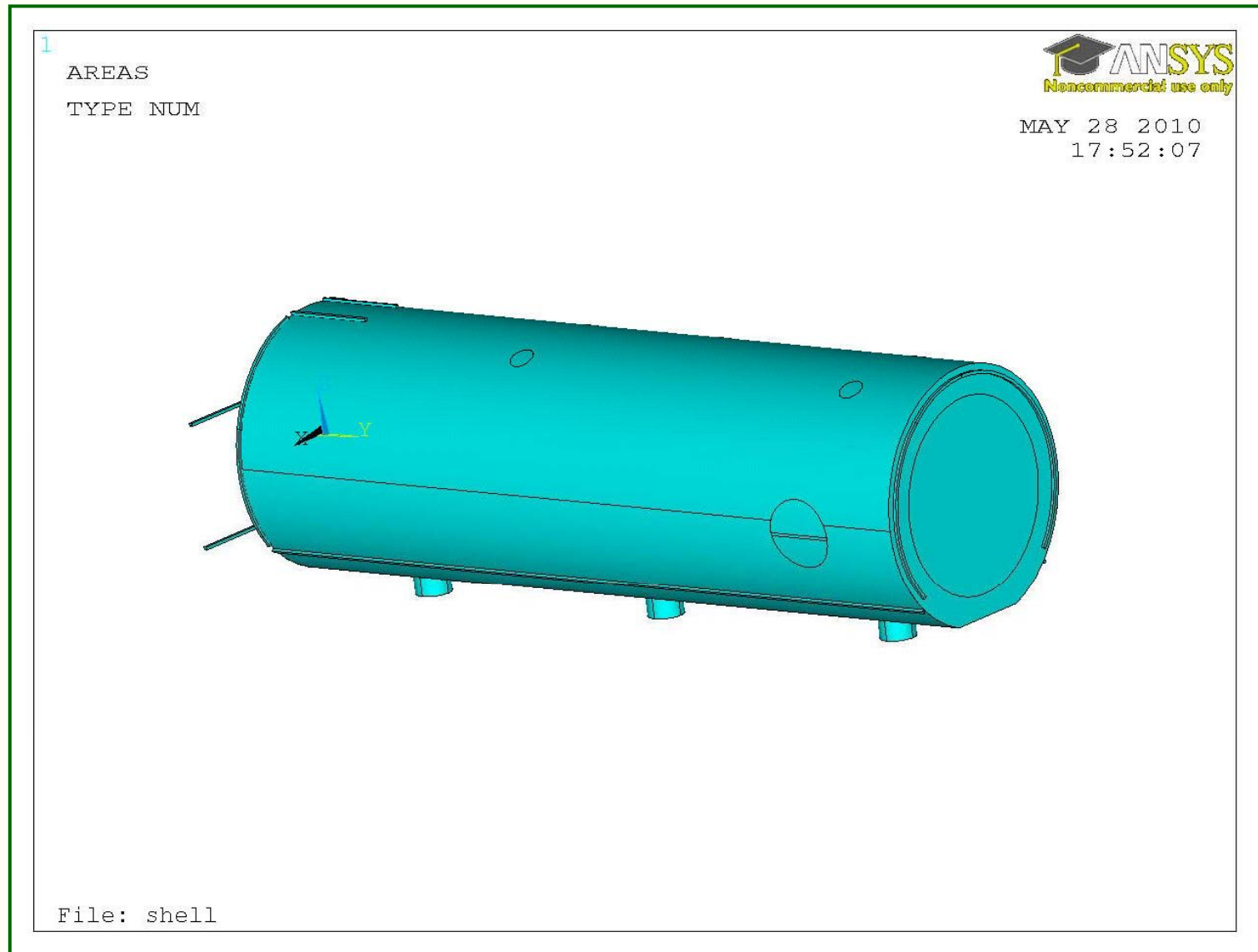
Based on this We have formulated our problem as

- Keep temperature of Cooling pipes initially at 80K
- Assume emissivity as $1.97e-3$
- Ignore the initial 25 minutes when cooling tube comes to 80K
- Keep cooling circuit
 - A. As in present HTS-1
 - B. As proposed for HTS-2
 - C. Based on the behavior of exercise A&B evolve our own scheme

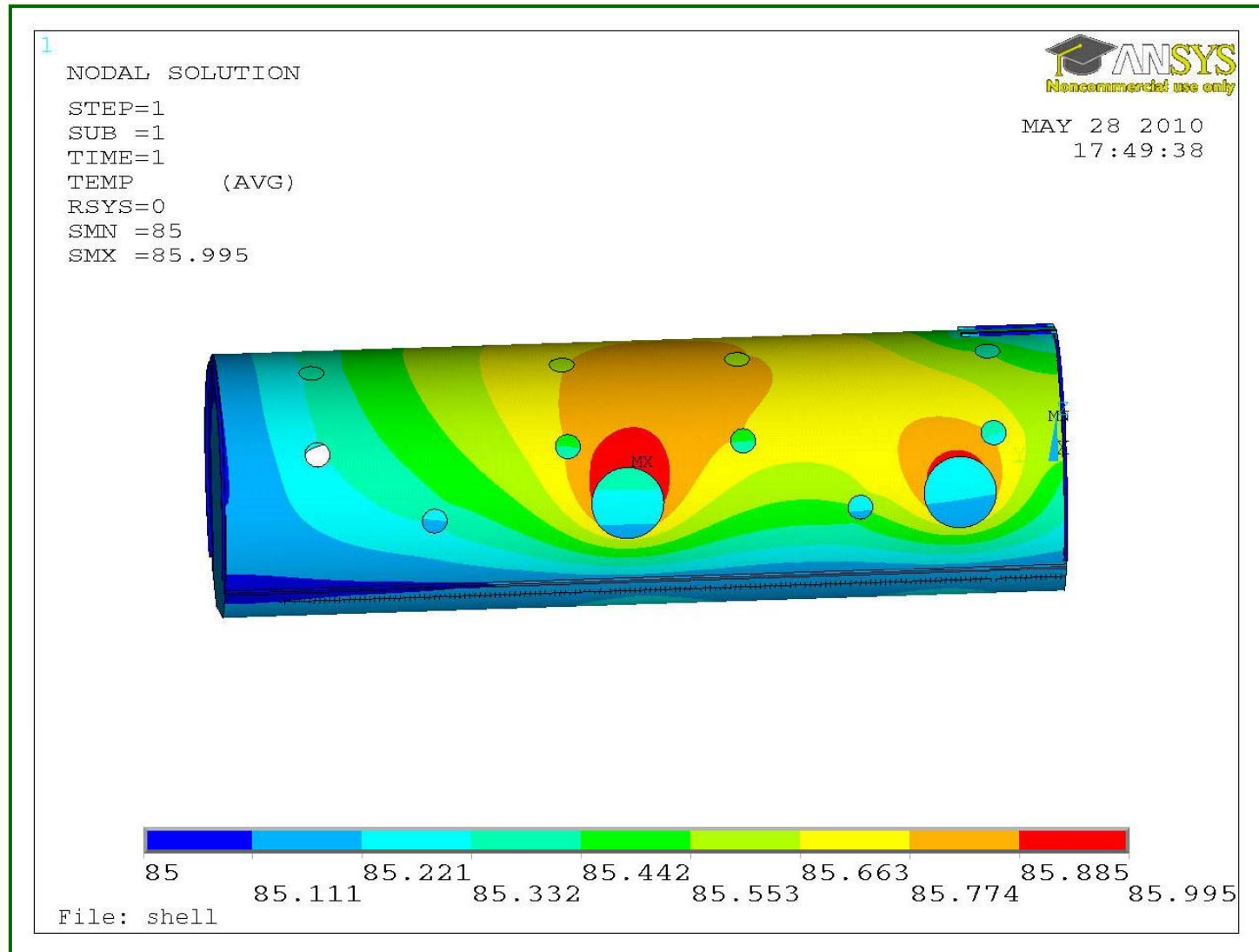
We want to find out

1. Steady State analysis to find out max. Temperature gradient.
2. Transient analysis to find out cool down time.
3. Transient analysis to find out maximum temperature gradient during cooling down.
4. Transient analysis to find out induced deformations.
5. Match the results with those available for HTS-1 or some other results.

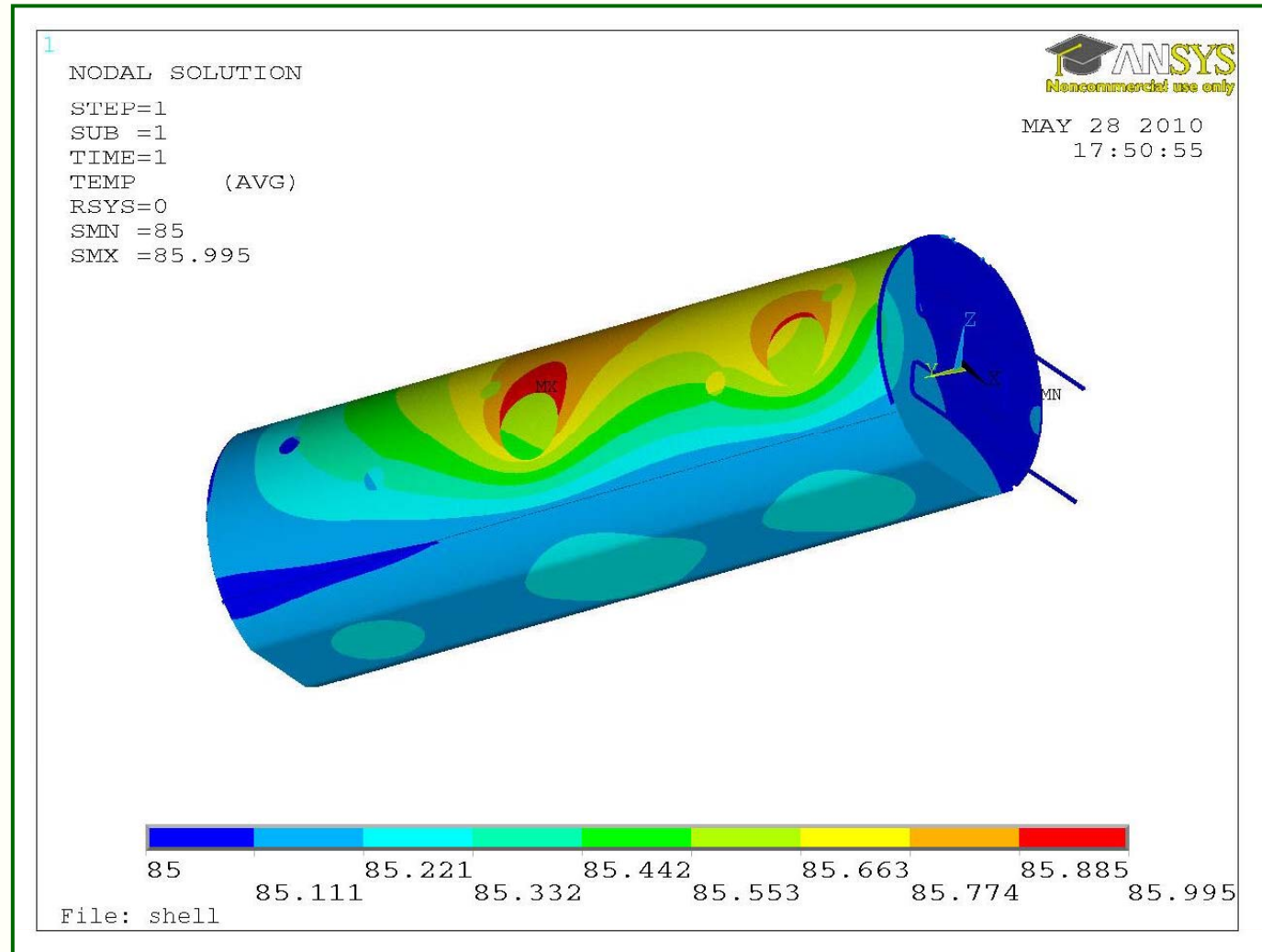
The 3-D Model of thermal Shield



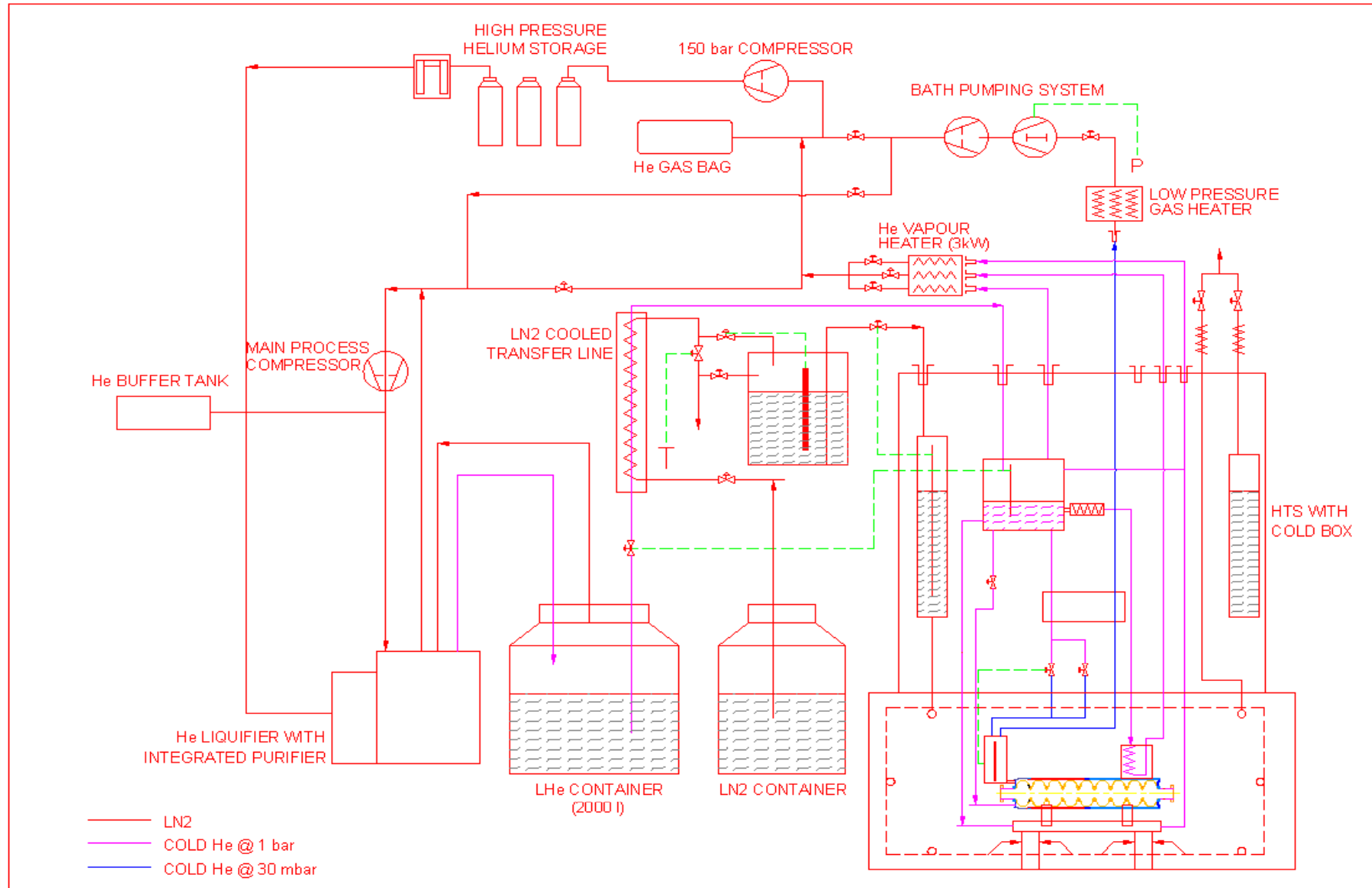
Steady State Analysis



Steady State Analysis



- **P & I Diagram for HTS**
- **Heat exchanger 4.5 K to 2K**
- **2K Supply system**
 - Although beam tubes of both the cavities will be isolated, but Cryogenically they may be tested jointly, to study cryogenic complications in case of CW operation.
- **Pressure Stability requirements during CW operation**
- **Retention of 4K thermal shield:** Now larger inventories of liquid helium is to be provided for CW testing. Also larger ratio of dynamic to static heat load is expected for CW testing. Can one think for omission of a 4 K thermal radiation shield?



- Time schedule of HTS cryostat looks extremely tight.
After finalization of drawings we will need minimum 18 months for fabrication. The HTS functional Requirement specifications document itself took 6 months. We can not go for fabrication till drawings are final.
- Need details of Cryogenic feed box (for size of port)
- Need details of cryogen requirements.
- Need to identify someone at FNAL end to begin RF power supply discussions.
- Need to identify someone at FNAL end to begin discussions on instrumentation and control.



Thank You