

# MQXFSD0 experience at CERN

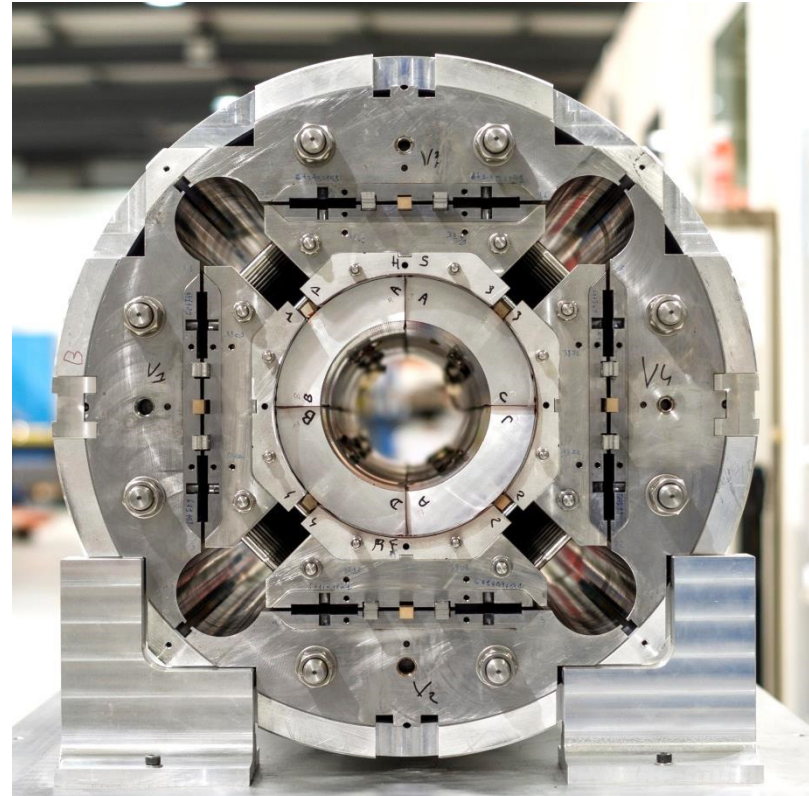
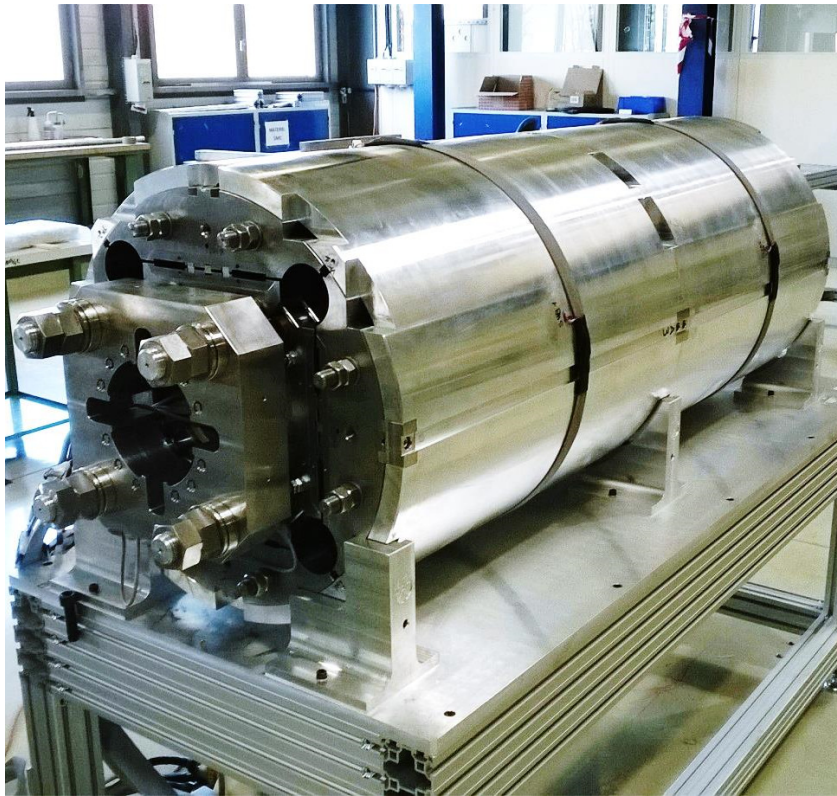
Mariusz Juchno, Hugo Bajas, Marta Bajko, Philippe Grosclaude, Michael Guinchard

# Outline

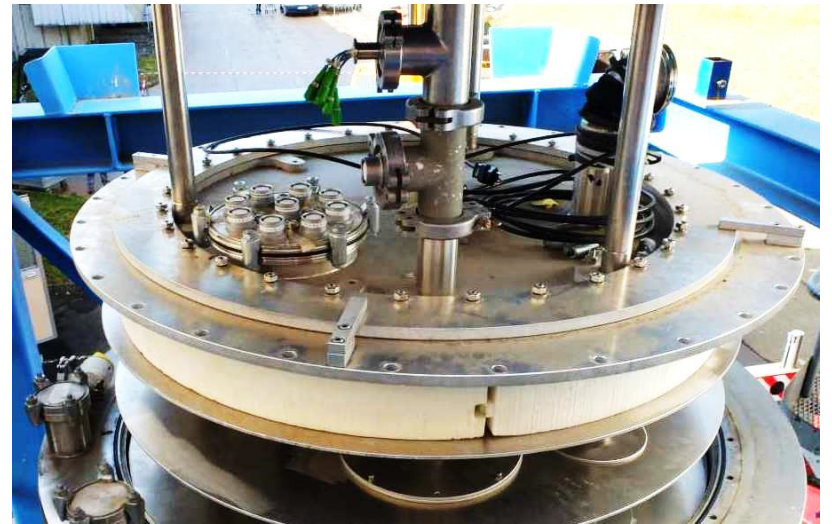
- Overview of the assembly and tests
- Coil-pack shims and dimensions
- Strain gauge location
- Details of the first RT assembly
- Details of the 77K tests
- Conclusions and next steps

# Assembled support structure with aluminium dummy coils

Two identical shell segments

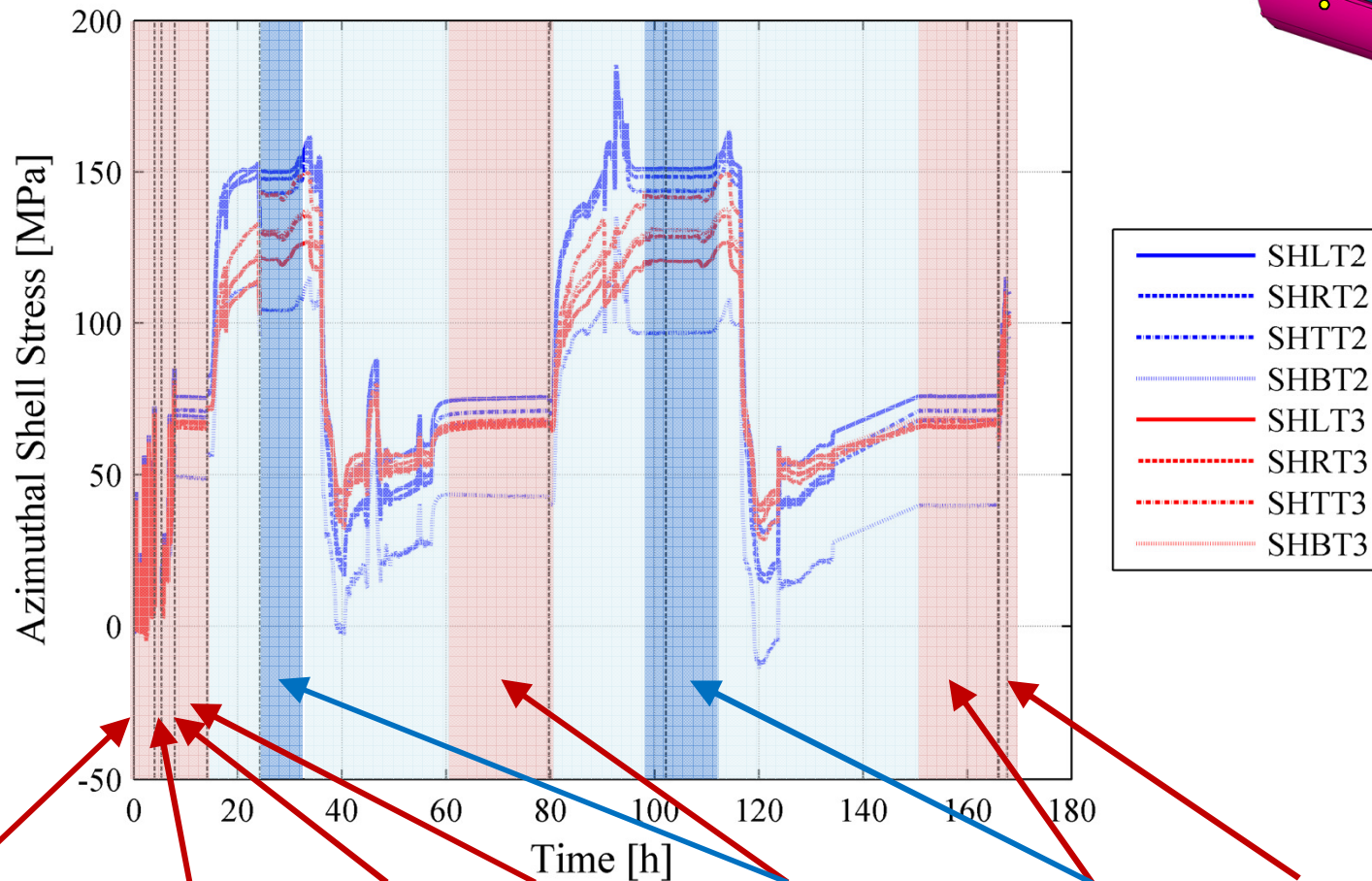
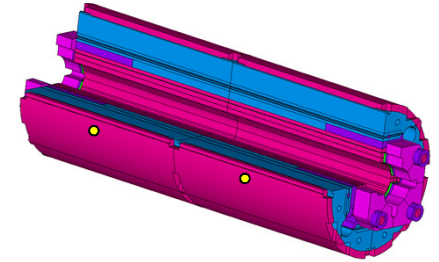


# Structure transported and tested in SM18



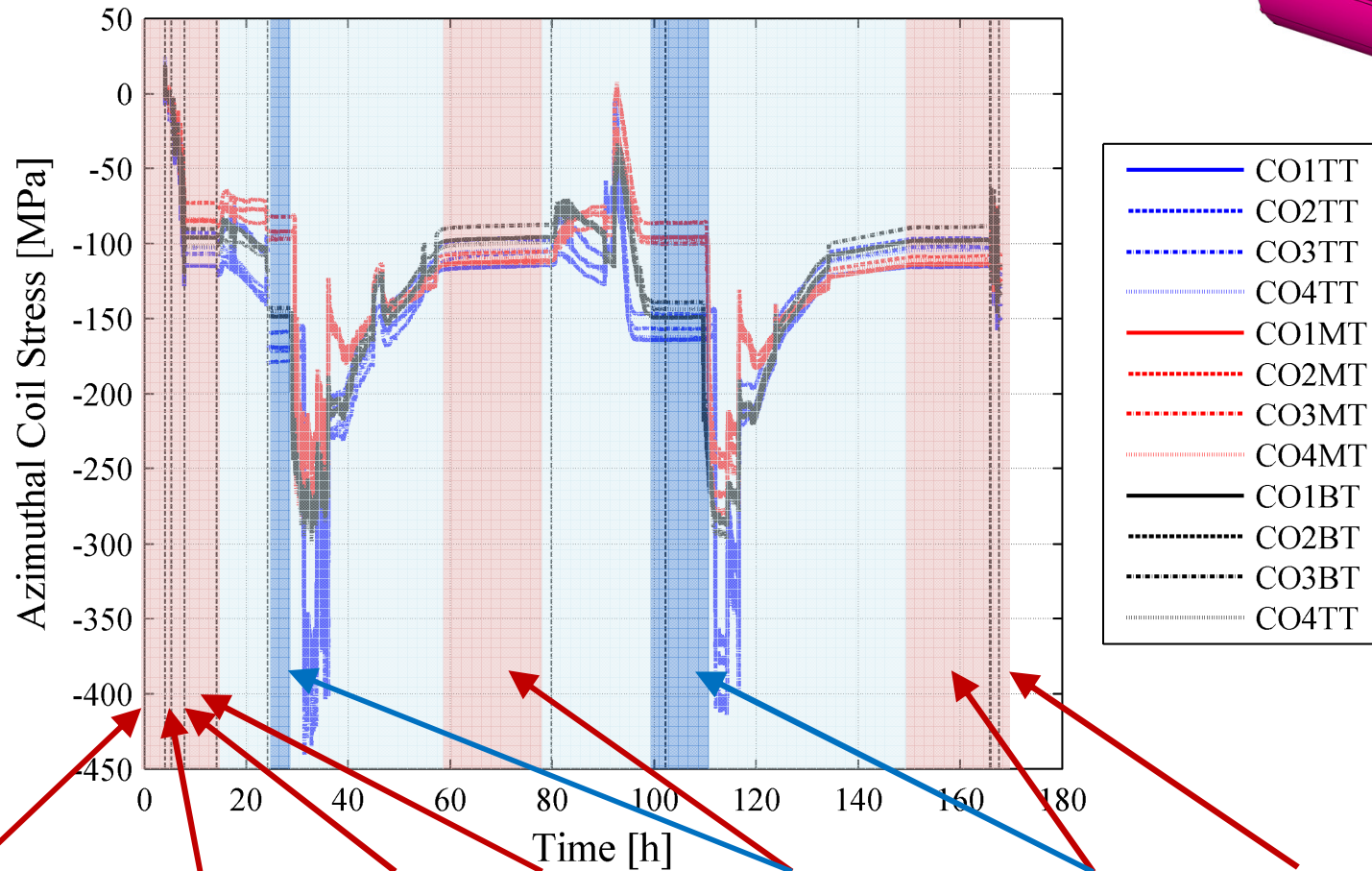
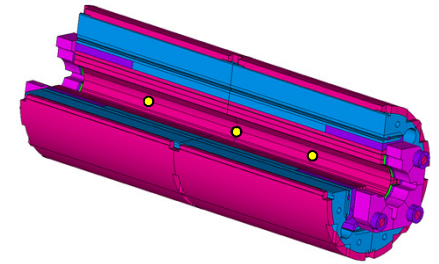
# Overview of the assembly and tests

## Shell azimuthal stress



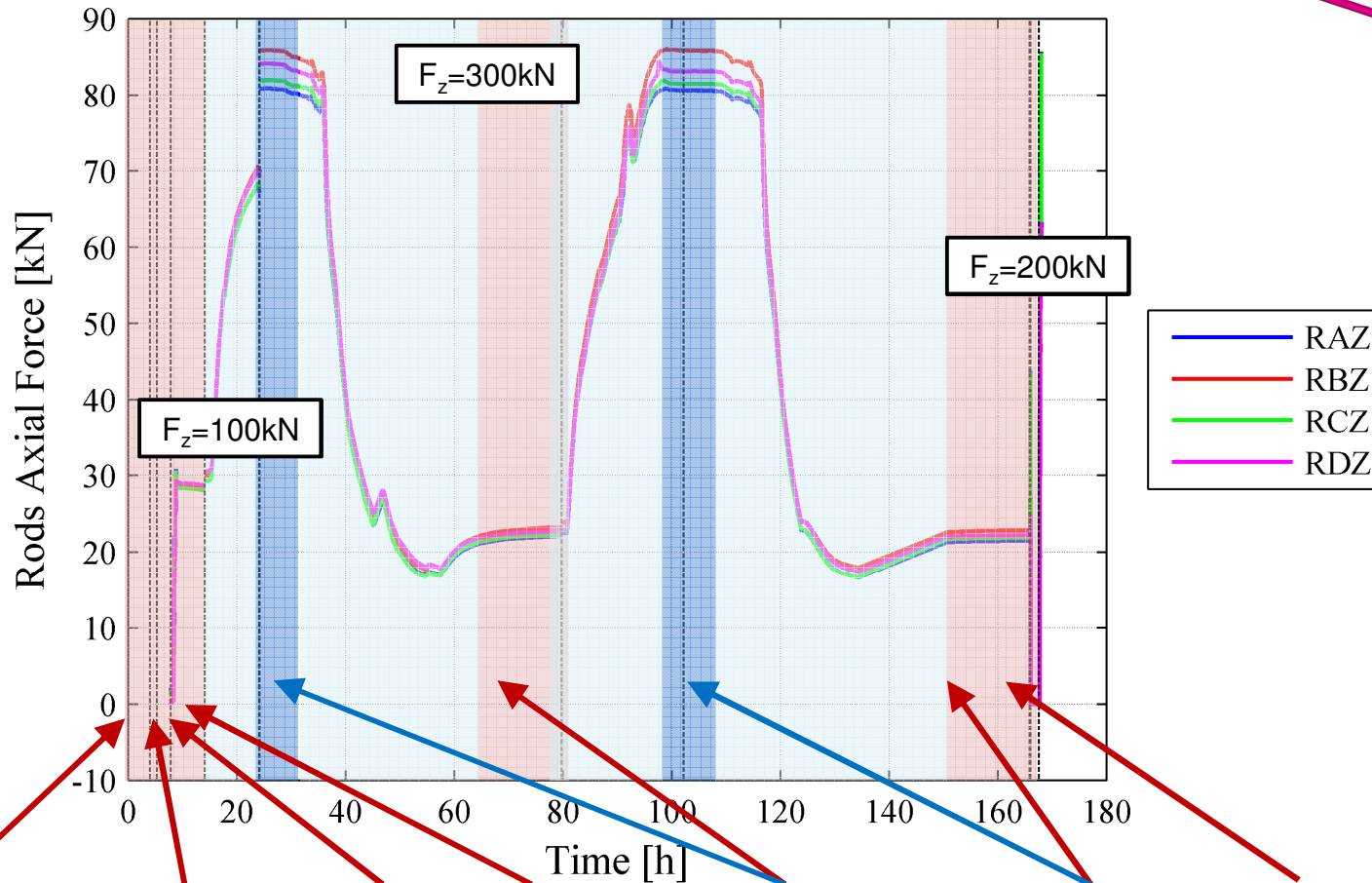
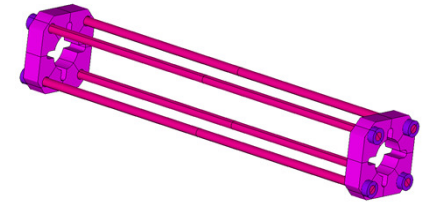
# Overview of the assembly and tests

## Coil azimuthal stress

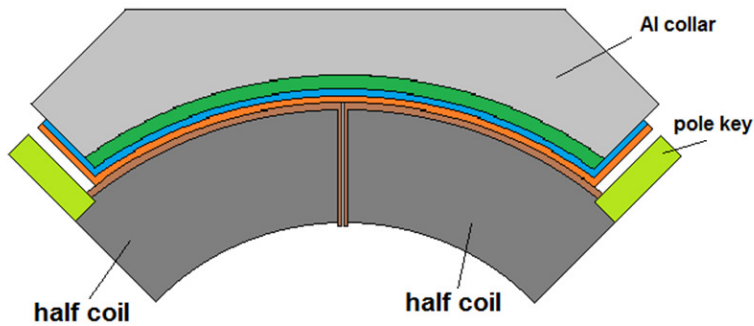


# Overview of the assembly and tests

## Rods axial force



# Coil-pack shims and dimensions



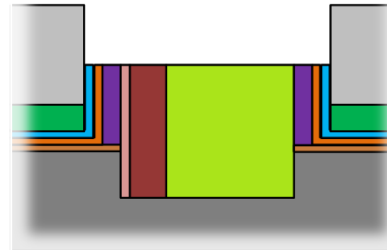
- kapton thk. 0.125 mm
- kapton thk. 0.125 mm
- kapton thk. 0.125 mm
- G11 0.5+0.5+0.2=1.2 mm

## Radial shims

- Nominal coil-collar distance 1.625 mm
- Inserted thk. 1.575 mm

## Coil-pack

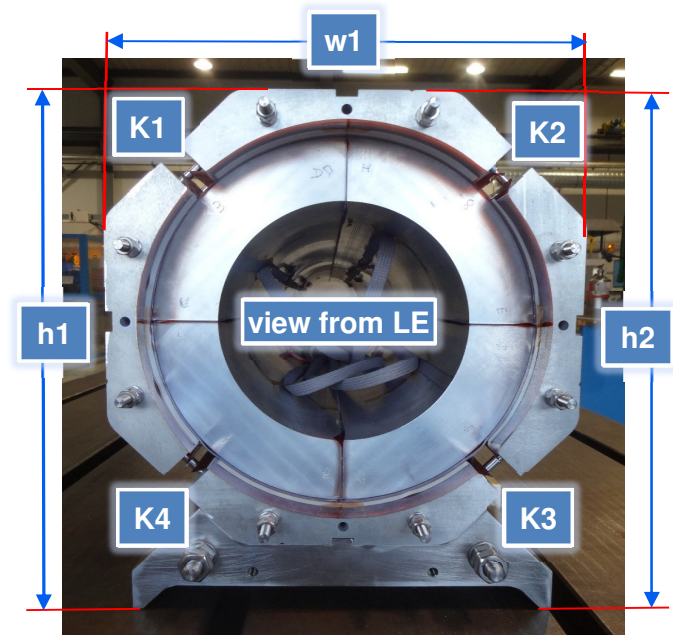
- Height within +/- 30 μm
- Width oversized by about 200 μm



- G10 12.75 mm
- Steel 1 mm
- Steel 0.2 mm
- Stainless 0.3 mm

## Pole-key

- Nominal collar distance 15 mm
- Inserted PK stack thickness 14.95mm



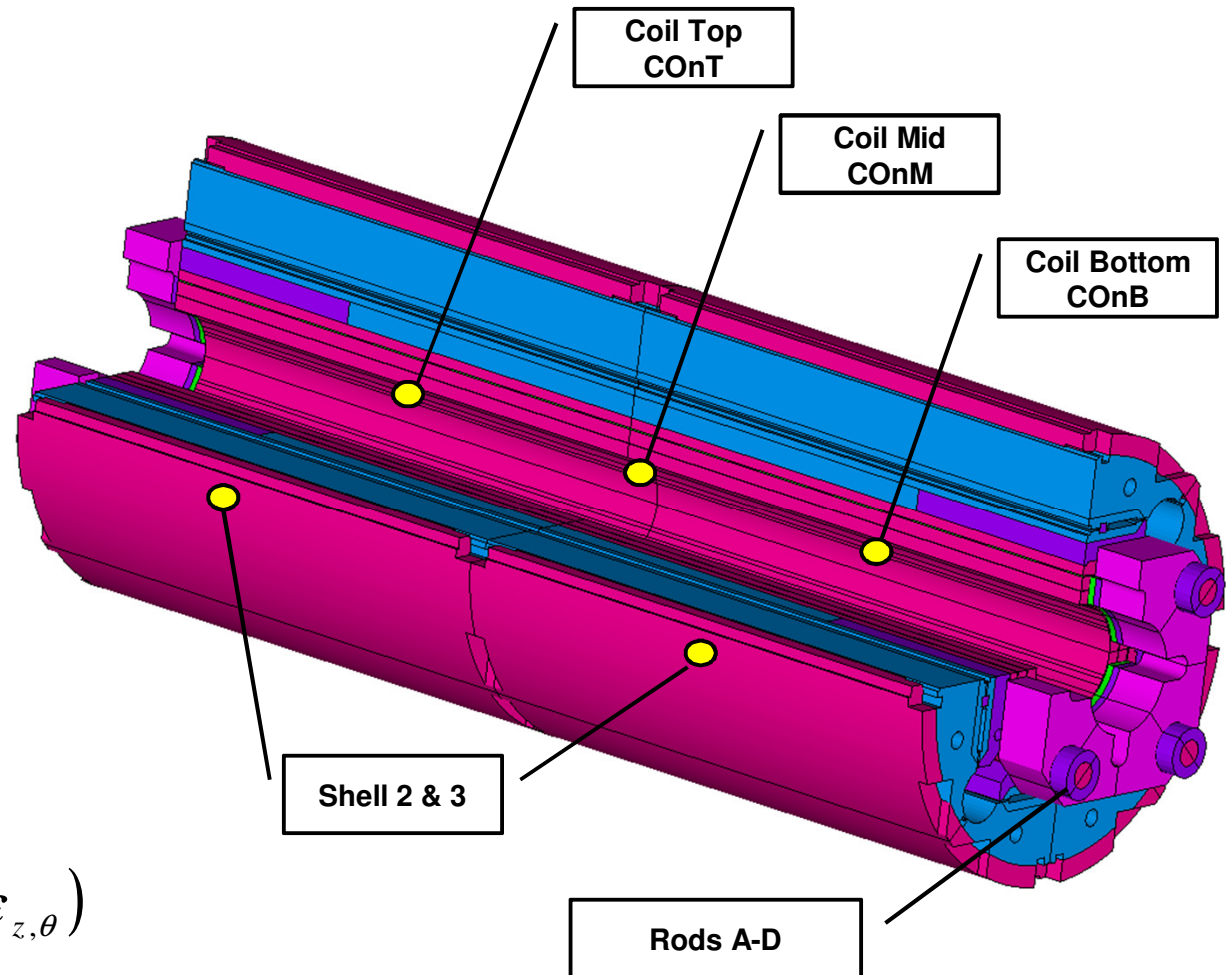
LE		x1	x2	x3	x4	x5	x6	x7	x8	Average	Nominal
		x (mm)	80	280	480	680	880	1080	1280	1480	-
	h1 (mm)	307.5	307.5	307.5	307.5	307.6	307.5	307.4	307.4	307.52	307.5
	h2 (mm)	307.5	307.5	307.5	307.4	307.5	307.4	307.4	307.4	307.47	307.5
	w1 (mm)	266.19	266.17	266.25	266.2	266.23	266.23	266.1	266.13	266.21	266

	AVG	STD
K1	14.95	0.022
K2	14.95	0.035
K3	15.01	0.036
K4	15.07	0.102
AVG	15.00	0.022



# Coils & Shells SG locations

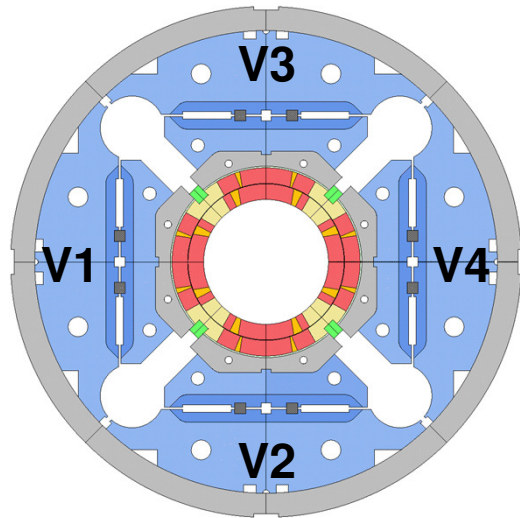
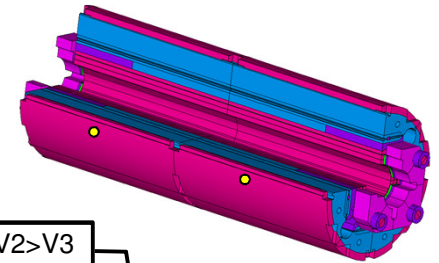
- 4 dummy coils
  - 3 stations on each (**T**op, **M**id, **B**ottom)
  - Azimuthal (**T**) and axial (**Z**) direction
- 2 shells
  - 4 stations on each (**L**eft, **R**ight, **T**op, **B**ottom)
  - Azimuthal (**T**) and axial (**Z**) direction
- 4 axial rods (**A**BC&**D**)
  - 1 station on each (full bridge)
  - Axial (**Z**) direction
- Stress estimation



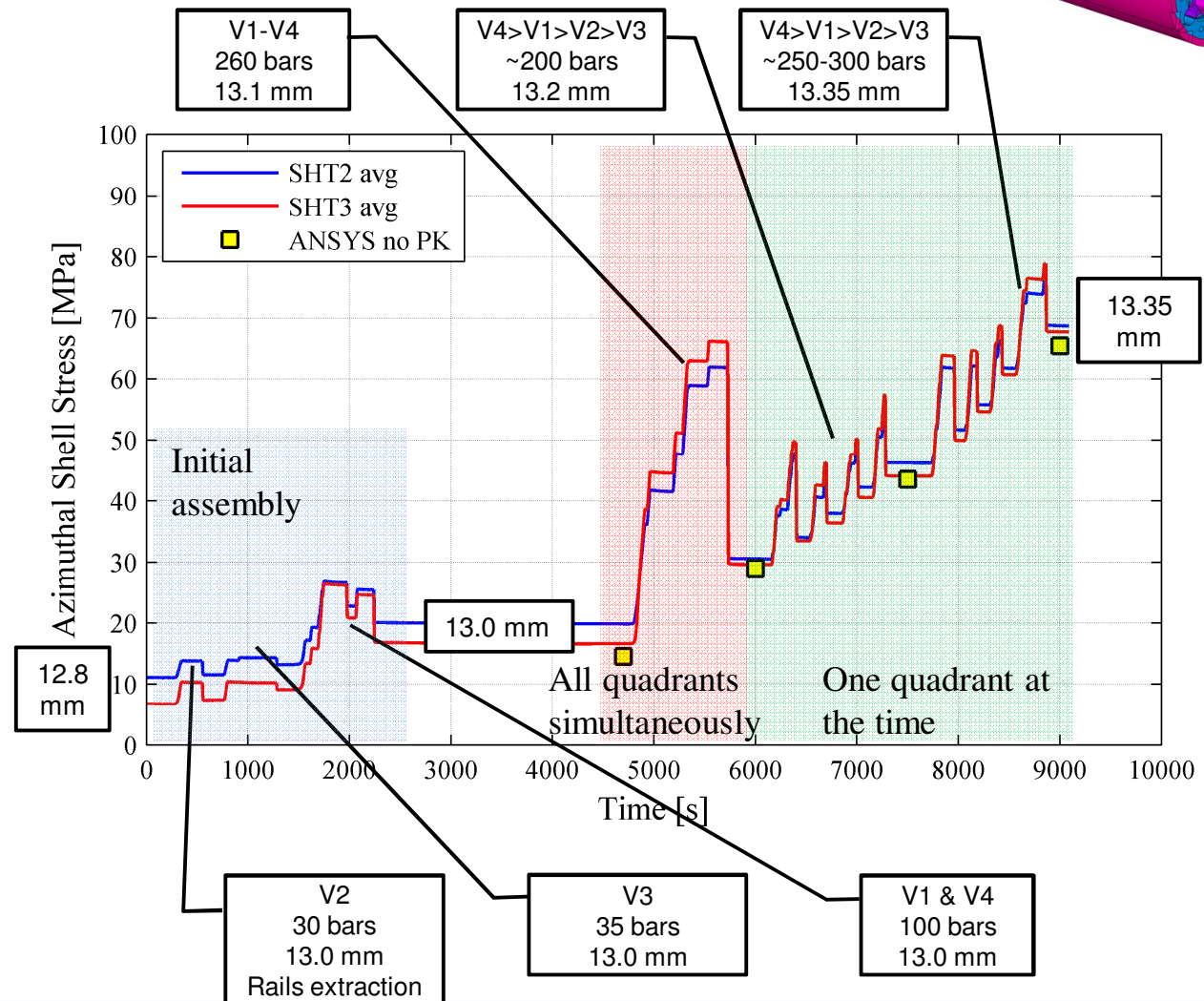
$$\sigma_{\theta,z} = \frac{E}{(1-\nu^2)} (\epsilon_{\theta,z} + \nu \epsilon_{z,\theta})$$

# Details of the first loading

## Shell azimuthal stress

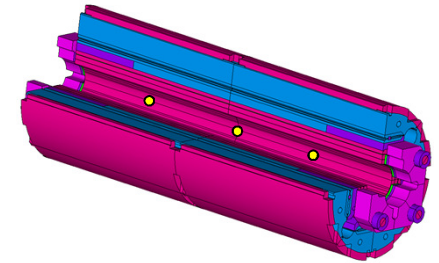


- Coil-pack oversized by about  $\sim 100\mu\text{m}$  per side
- Effective shim of  $450\mu\text{m}$

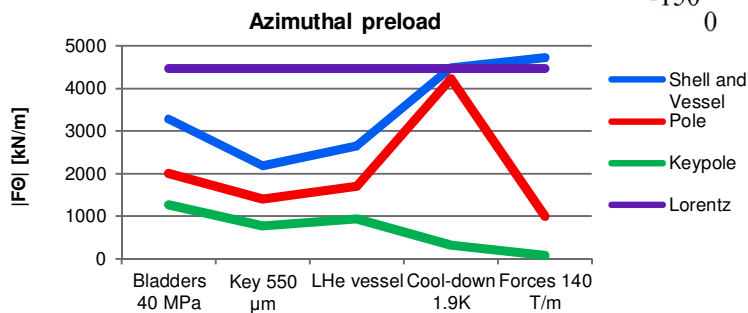
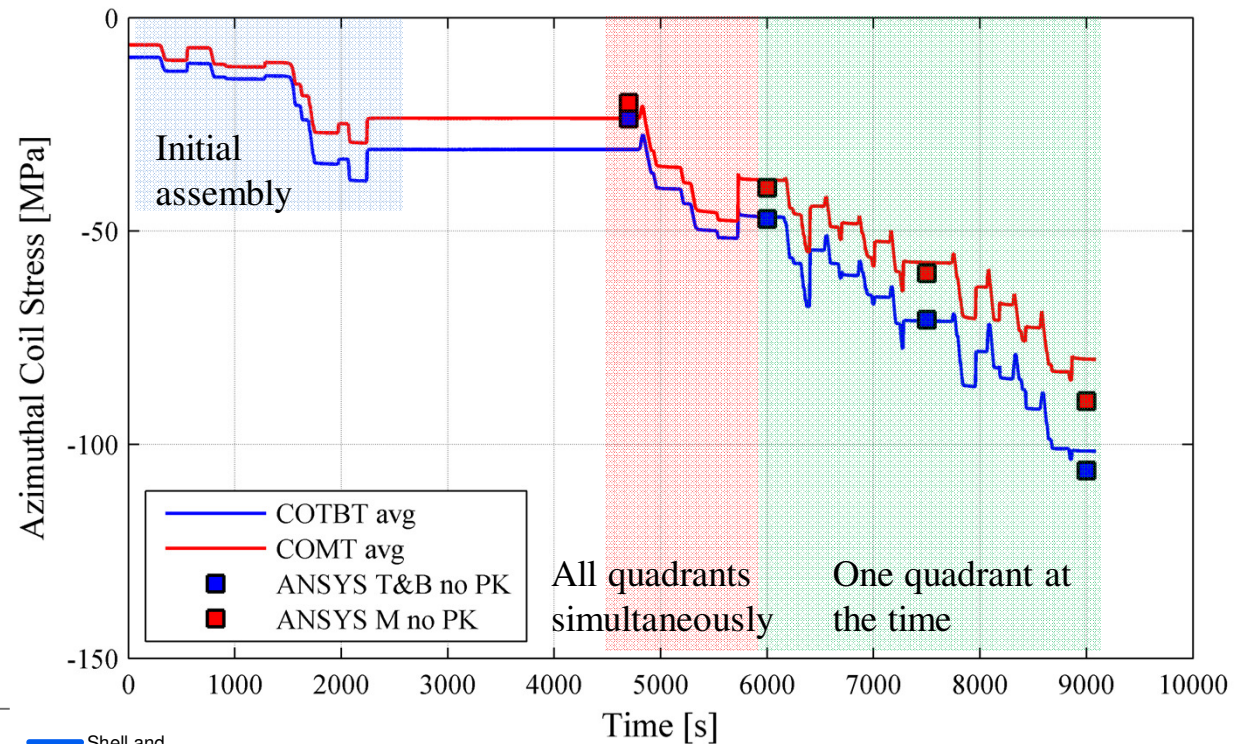
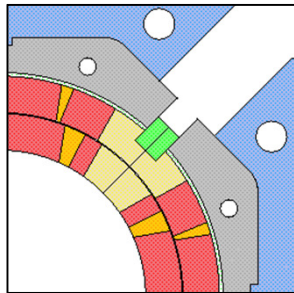


# Details of the first loading

## Coil azimuthal stress

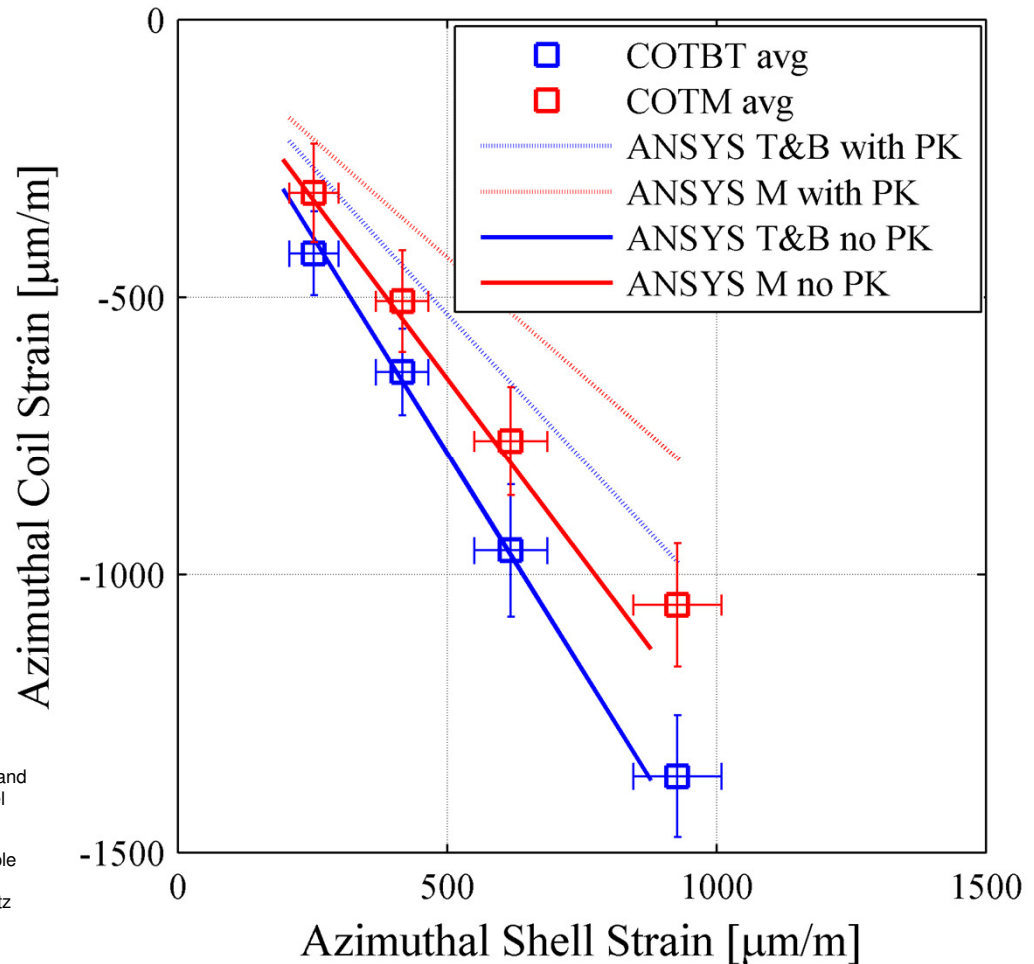
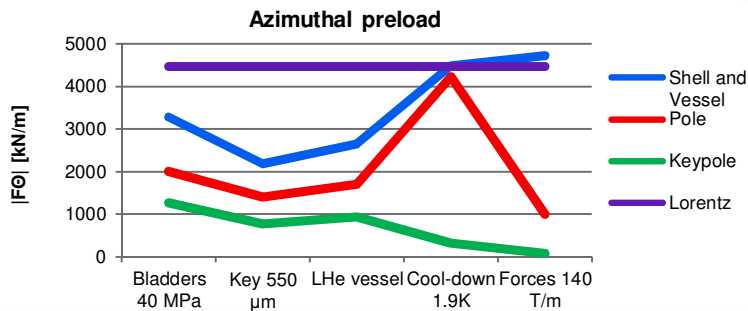
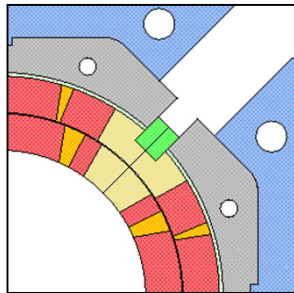


- ANSYS simulation performed with and without the polekey
- Measurement indicate that the polekey does not intercept the force during assembly

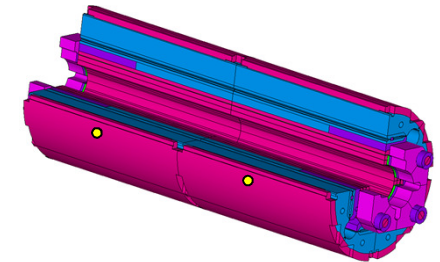
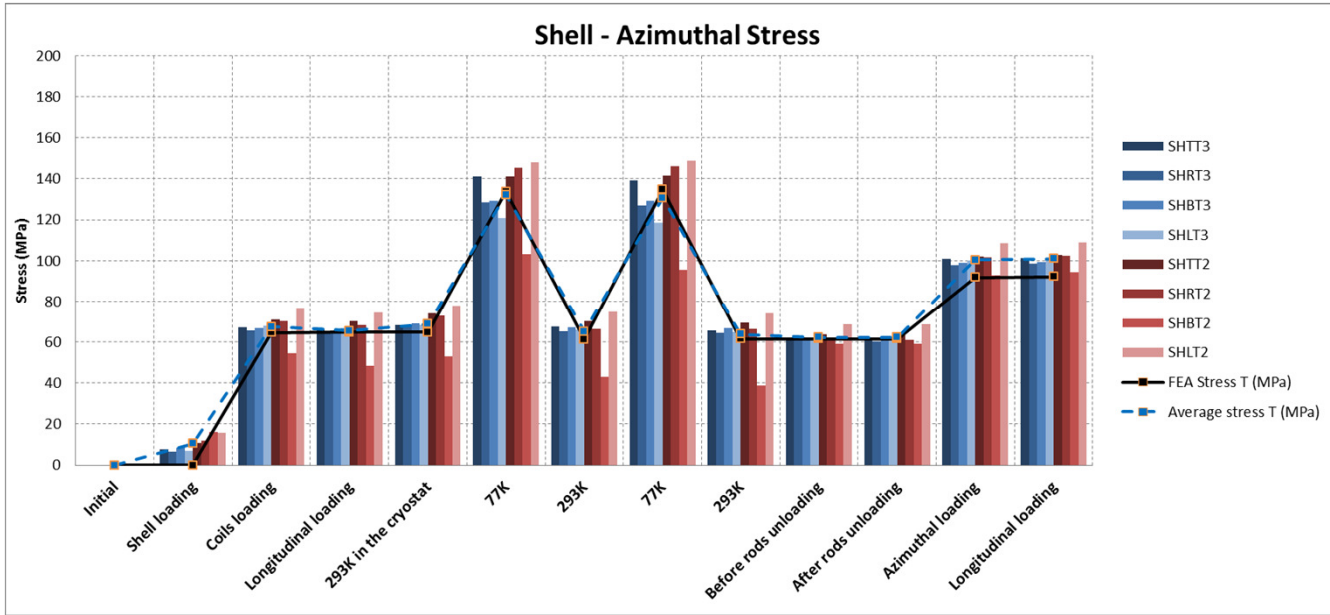


# Shell vs Coil strain

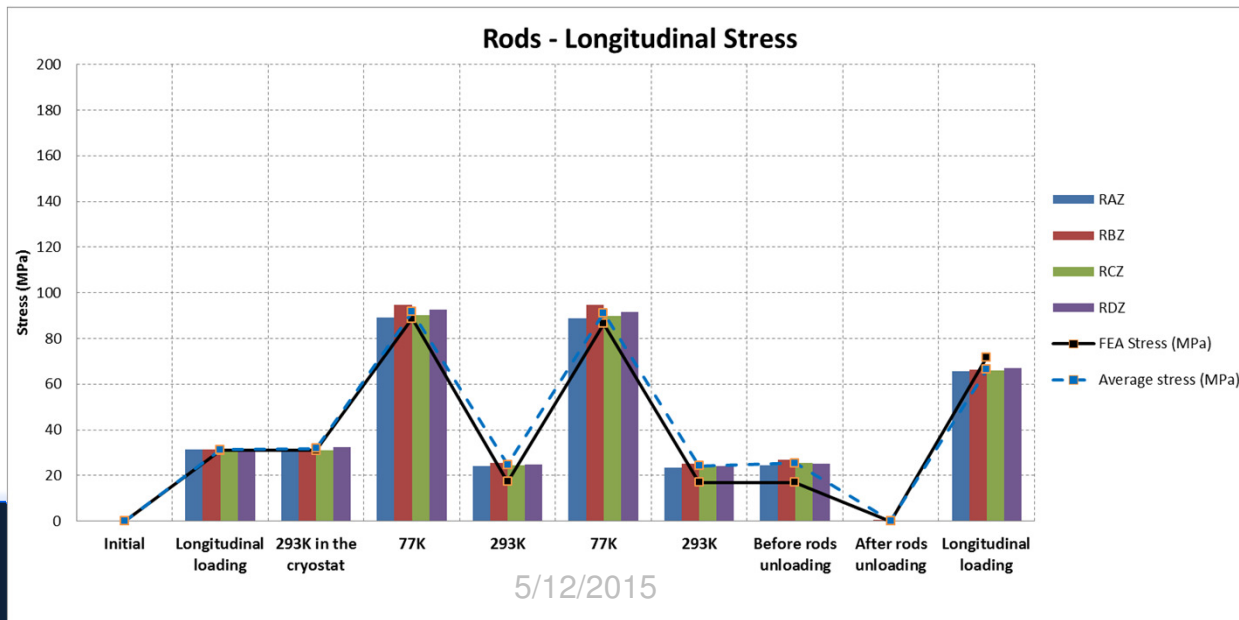
- ANSYS simulation performed with and without the polekey
- Measurement indicate that the polekey does not intercept the force during assembly



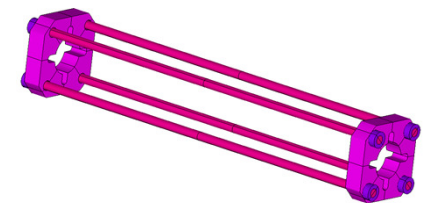
# Stress in the shell and axial rods



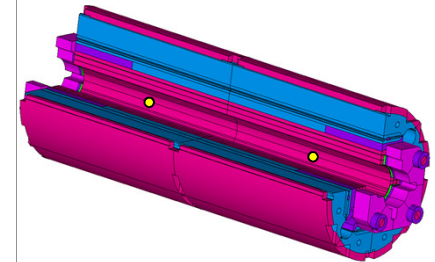
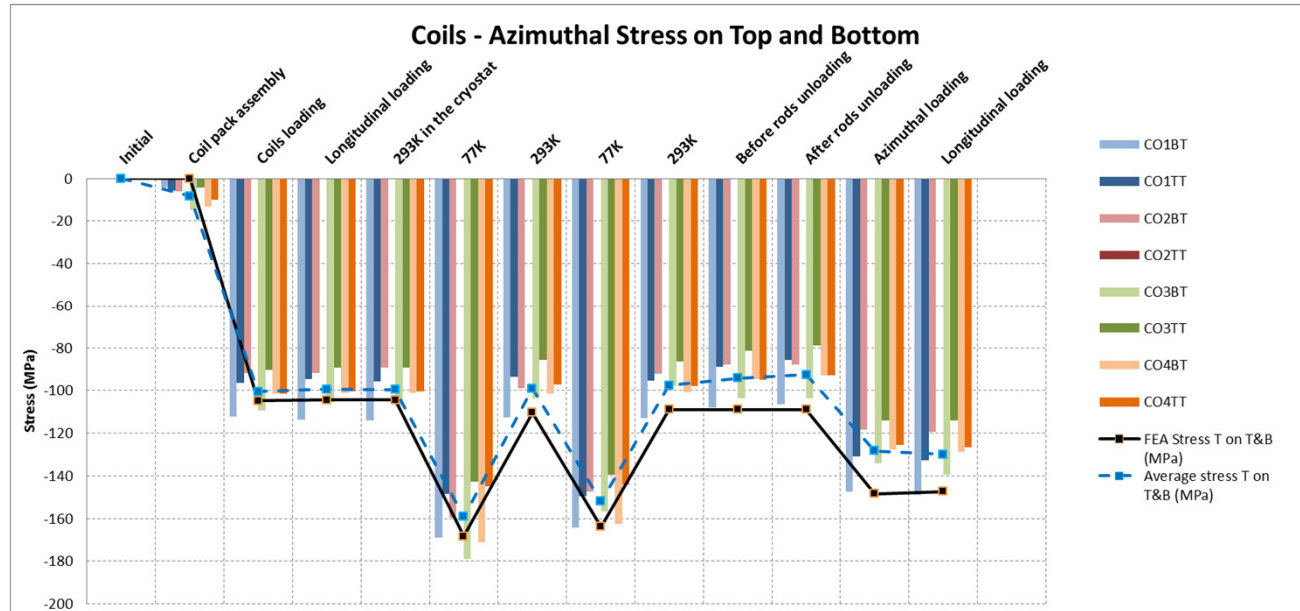
Mechanical elements providing the preload



Good consistency between the model and the SG data

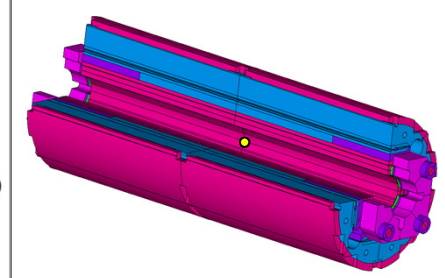
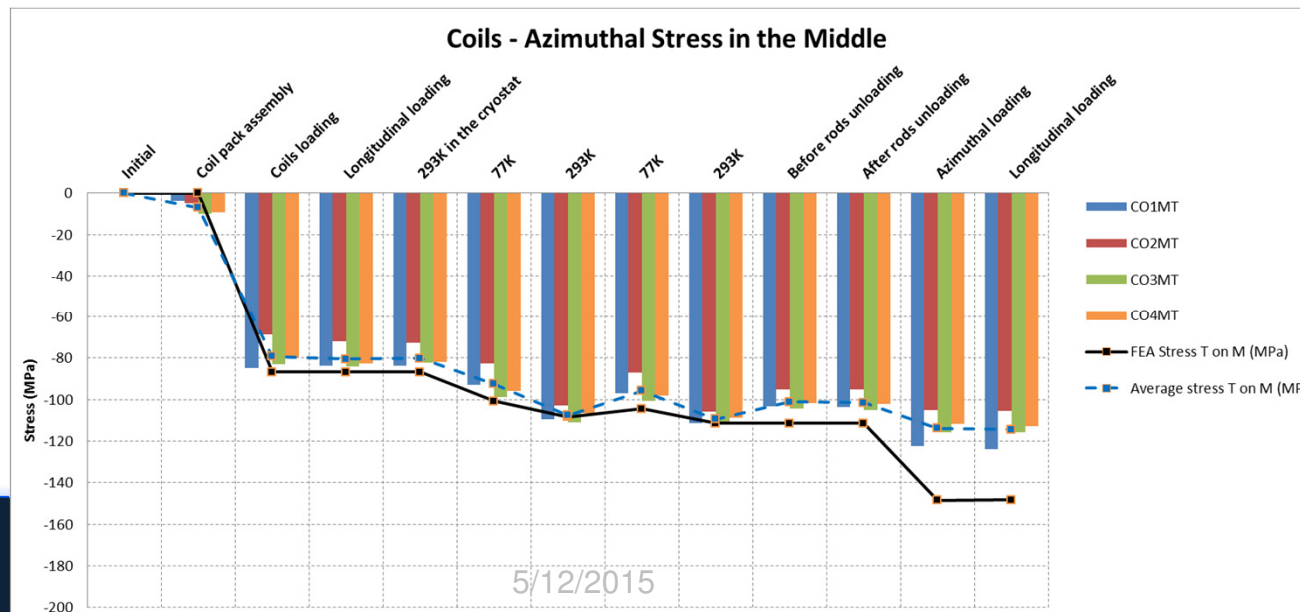


# Azimuthal coil stress



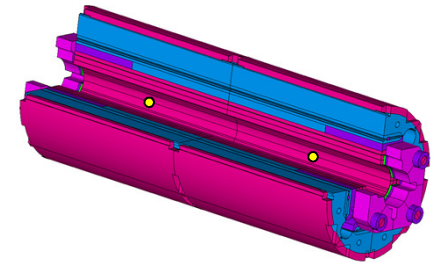
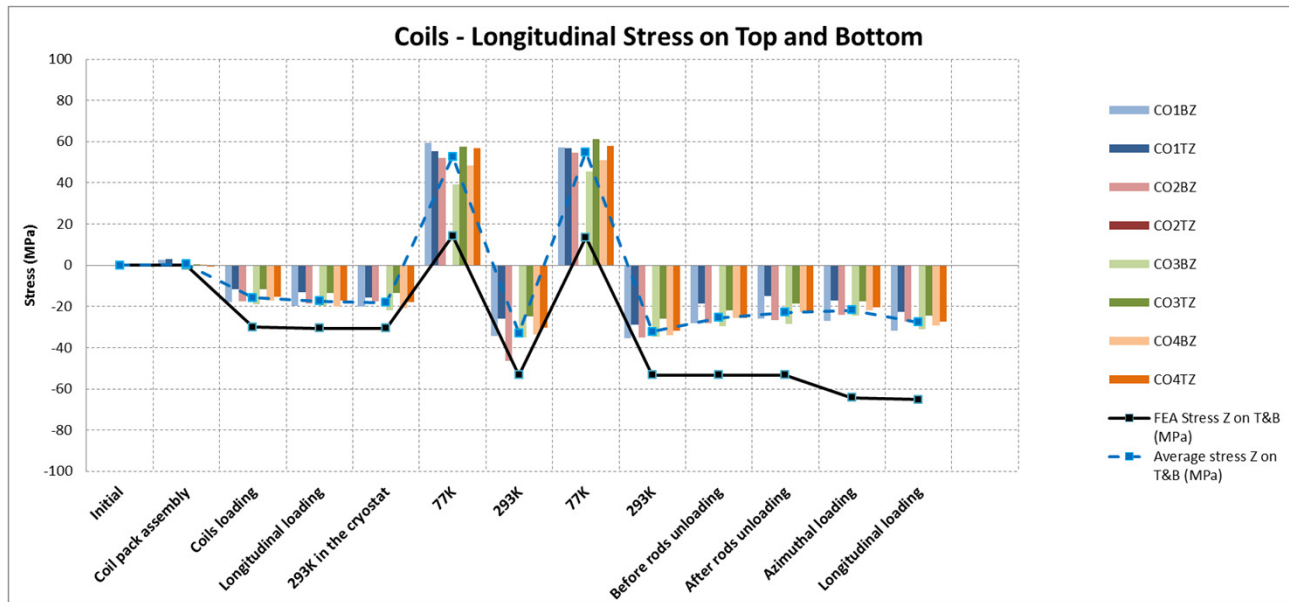
Good consistency for the T&B location

M location more difficult due to shells interface and friction (but overall well modelled)



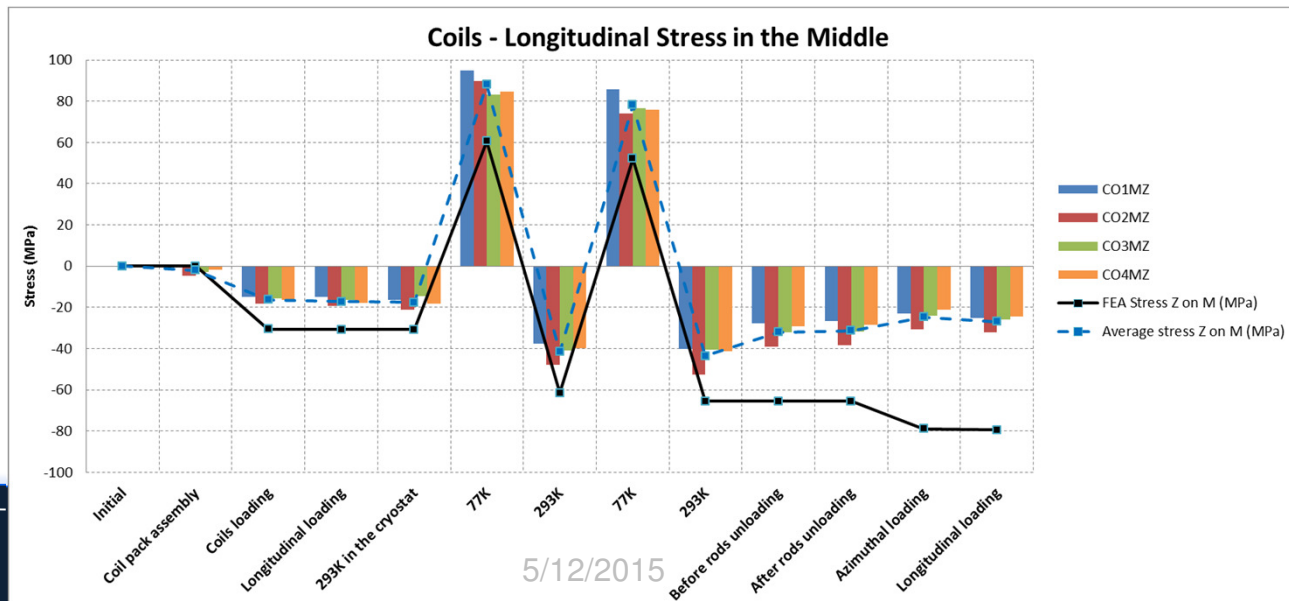
5/12/2015

# Axial coil stress

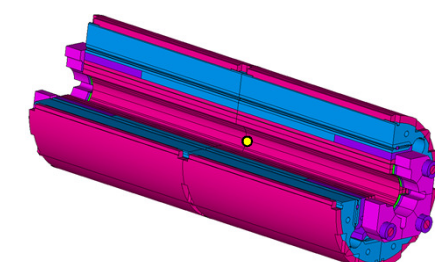


Axial direction generally more difficult due to friction

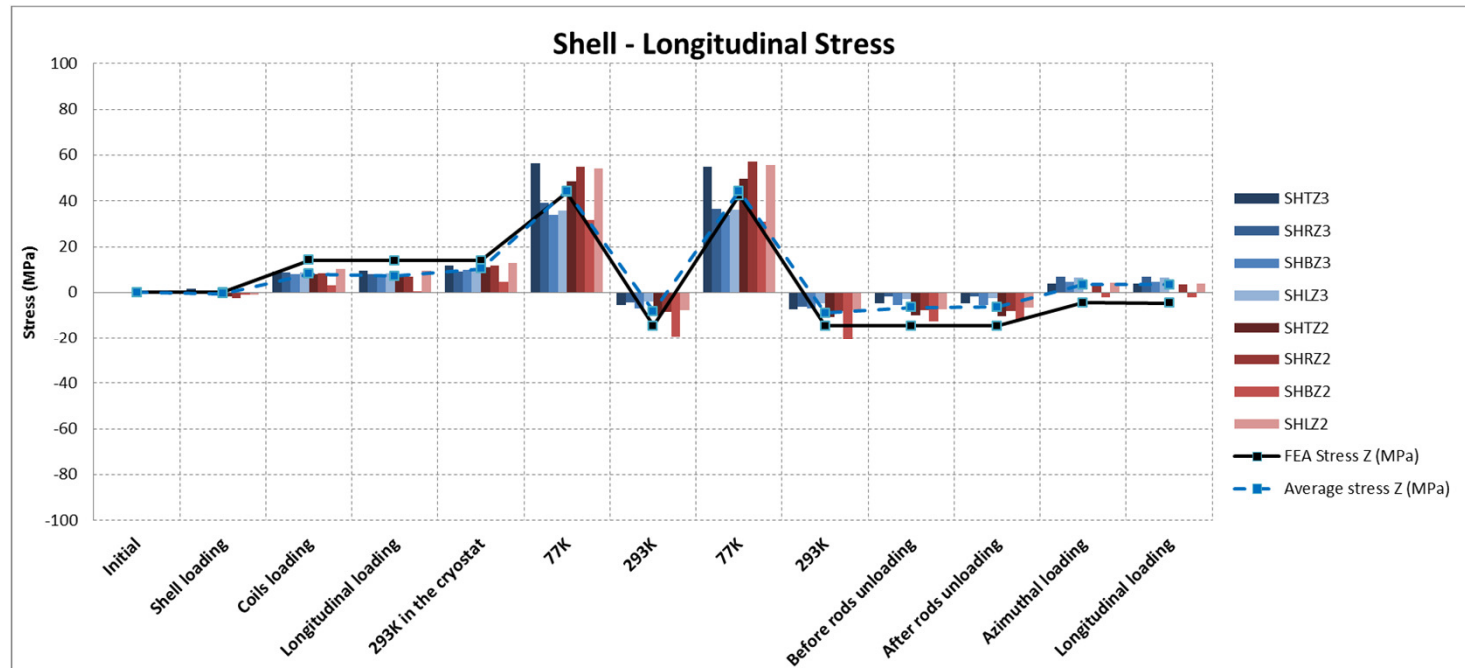
With  $f=0.3$  the overall behavior is modelled properly



Friction effect still being investigated



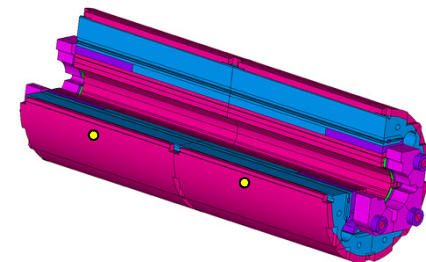
# Axial shell stress



Path dependent effects introduced by friction

Numerical analysis performed with multiple sub-steps

Fine-tuning of the friction coefficient (from 0.2 to 0.3)





# Conclusions

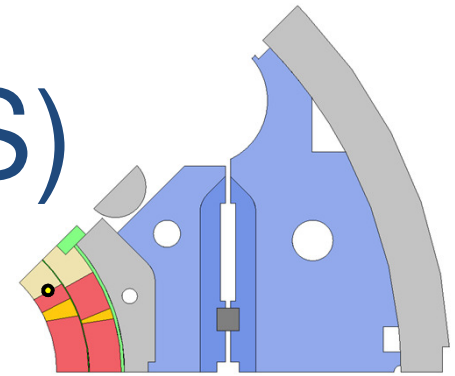
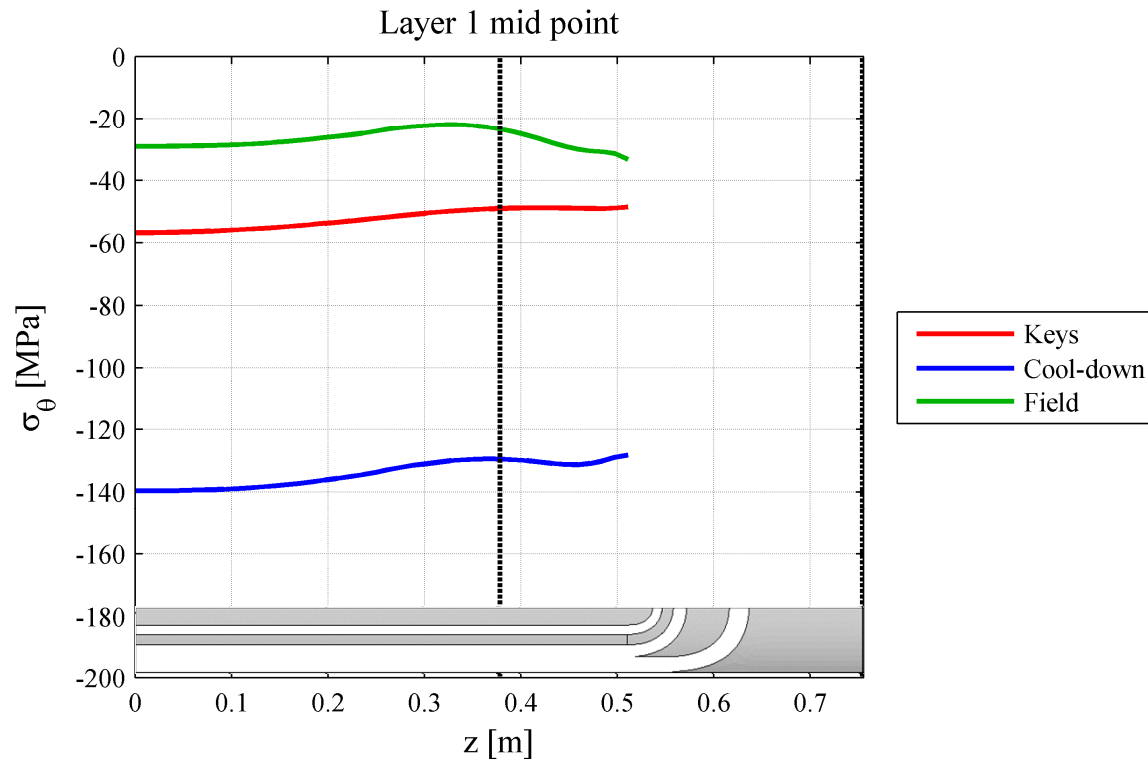
- Very good results which agree with the numerical prediction.
- Successful validation and fine-tuning of the numerical model of the structure

## Next steps

- Third and fourth thermal cycle of the present model ongoing
- Disassembly and second assembly with dummy coils and the new set of shells  
(optimized design to decrease the stress variation along the coil and decrease the stress in the coil head)

# Appendix

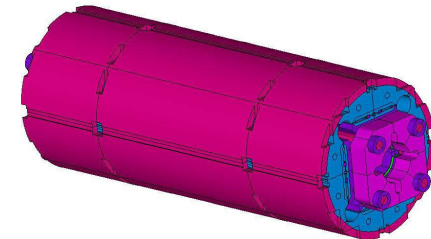
# Stress variation (MQXFS)



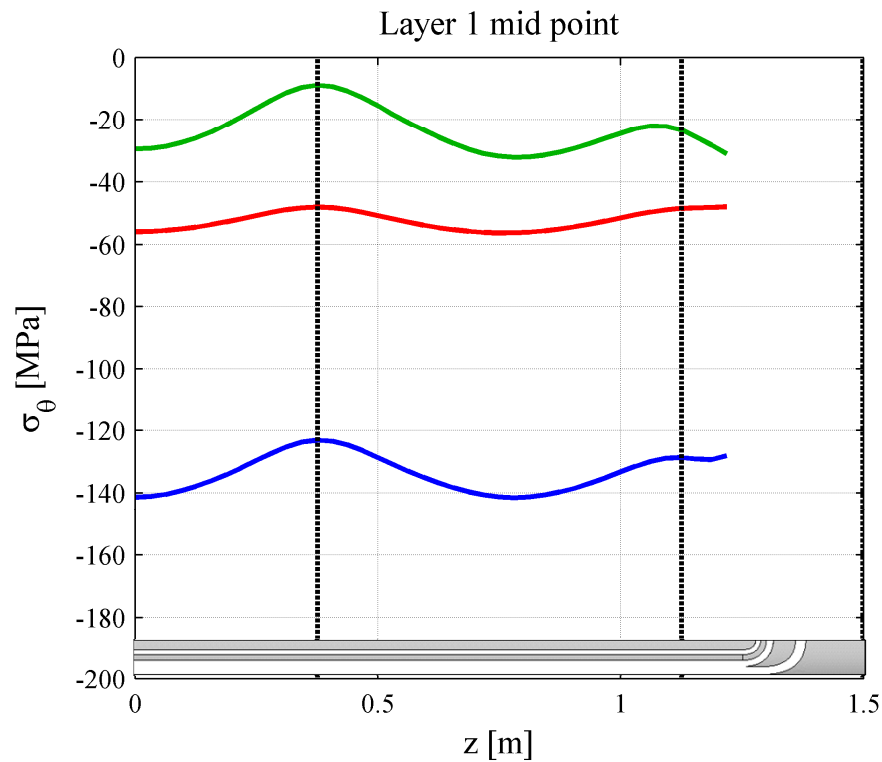
- Preload guaranteed at 140 T/m
- Stress variation under central segment **+/- 5 MPa** (affected by ends)
- Half-shells at the extremities decrease the stress variation

## • Optimised layout

- 1 normal shell (0.755m long)
- 2 half-shells at extremities (0.377m)

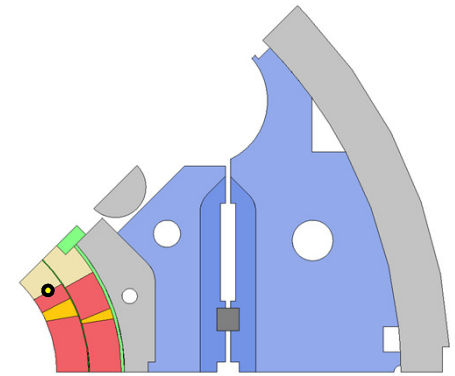


# Stress variation (3m)



- **Optimised layout**

- 3 normal shell (0.755m long)
- 2 half-shells at extremities (0.377m)



- Stress variation under central segments **+/- 10 MPa**
- Half-shells at the extremities decrease the stress variation (affect adjoining long segments)

